

## Citation

Li, Z. and Yang, Q. and Fang, R. and Chen, W. and Hao, H. 2021. Origami metamaterial with two-stage programmable compressive strength under quasi-static loading. *International Journal of Mechanical Sciences*. 189: ARTN 105987.  
<http://doi.org/10.1016/j.ijmecsci.2020.105987>

# 1 Origami metamaterial with two-stage programmable 2 compressive strength under quasi-static loading

3 Zhejian Li<sup>1</sup>, Qiusong Yang<sup>2</sup>, Rui Fang<sup>2</sup>, Wensu Chen<sup>1\*</sup>, Hong Hao<sup>1\*</sup>,

4 <sup>1</sup>*Centre for Infrastructural Monitoring and Protection, School of Civil and Mechanical Engineering,*  
5 *Curtin University, Australia*

6 <sup>2</sup>*School of Civil Engineering, Guangzhou University, China*

7 \*Corresponding authors: [wensu.chen@curtin.edu.au](mailto:wensu.chen@curtin.edu.au) (W. Chen); [hong.hao@curtin.edu.au](mailto:hong.hao@curtin.edu.au) (H.  
8 Hao)

## 9 Abstract

10 An origami metamaterial with two-stage programmable compressive strength is proposed by  
11 combining the stacked Miura-origami and rhombic honeycomb structure. By adjusting the  
12 geometries of the structure, the compressive response of each stage including the compressive  
13 strength and the densification strain can be programmed within a certain range. Furthermore,  
14 the initial peak force, as an undesired energy-absorbing characteristic, can be programmed to  
15 maintain at a low level. The commonly seen fluctuation of crushing resistance on honeycomb  
16 structure is also minimized during the second stage deformation. The crushing behaviour of  
17 origami metamaterial is investigated under quasi-static loading condition. The  
18 programmability of compressive properties is demonstrated for the two stages of the  
19 deformation. The analytical model of the two-stage compressive response of the proposed  
20 origami metamaterial is firstly developed with friction contribution being taking into  
21 consideration during the first deformation stage. The analytical model is then verified with  
22 numerical analysis and quasi-static compressive testing data. The programmability of its  
23 compressive properties such as the initial peak crushing resistance, mean crushing force for  
24 both stages of deformation are then analysed based on the verified analytical model.

25

26 **Keywords:** Origami metamaterial; two-stage compressive strength; programmable  
27 compressive properties; quasi-static loading

28

Nomenclature	
$a, b$	length of edges on the top section of the unit column
$h_1, h_2$	height of half a Miura-ori cell and height of a rhombic tube cell
$h_{1,0}, h_{1,f}$	the initial and final height of half a Miura-ori cell
$H$	overall height of the structure
$W, L$	width and length of the unit column
$W_0, W_f$	the initial and final width of the unit column
$L_0, L_f$	initial and final length of the unit column
$m, n$	number of unit column in $X_1$ and $X_2$ direction
$x, y$	number of Miura-ori cell and rhombic tube cell in the unit column
$T$	cell wall thickness
$\alpha$	the angle of $A_2A_1B_1$ on the top section of the Miura-ori cell
$\beta$	the angle between two faces on the top section of the Miura-ori cell
$\theta$	the angle between faces of Miura-ori cell and vertical plane
$\gamma$	the angle between faces of Miura-ori cell and rhombic tube cell
$\varphi$	the angle between faces of rhombic tube cell
$A_{sweep}$	the total swept area of bottom edges on base throughout crushing
$\mu$	friction coefficient
$P_{m1}, P_{m2}$	mean crushing force of the structure during the 1 <sup>st</sup> and 2 <sup>nd</sup> stage of deformation with friction considered
$F$	crushing force of the unit column during the 1 <sup>st</sup> stage of deformation without considering the friction resistance
$F_{m1}$	mean crushing force of the unit column during the 1 <sup>st</sup> stage of deformation without considering the friction resistance
$F_{m1(m \times n)}$	mean crushing force of metamaterial with $m \times n$ unit column without considering the friction resistance
$F_c$	critical crushing force during the 1 <sup>st</sup> stage of deformation
$F_f$	average friction force on bottom edges during the 1 <sup>st</sup> stage of deformation
$F_{mf}$	mean friction contributed crushing force during the 1 <sup>st</sup> stage of deformation
$E_b$	bending energy of the Miura-ori section of the unit column during the 1 <sup>st</sup> stage of deformation
$E_f$	total friction energy from the interfaces during the 1 <sup>st</sup> stage of deformation
$F_{m2,I}, F_{m2,II}, F_{m2,III}$	mean crushing force of each Type I, II, III element in rhombic honeycomb section during 2 <sup>nd</sup> stage of deformation
$M_0$	plastic bending moment per unit length
$\sigma_0$	yield strength of the base material
$\kappa$	coefficient of effective crushing distance during 2 <sup>nd</sup> stage of deformation
$\rho_0$	the density of the base material
$\rho_r$	relative density of the structure
$\varepsilon_D, \varepsilon_{D1}, \varepsilon_{D2}$	overall densification strain, densification strain of the 1 <sup>st</sup> and 2 <sup>nd</sup> stage of deformation

## 30 **1 Introduction**

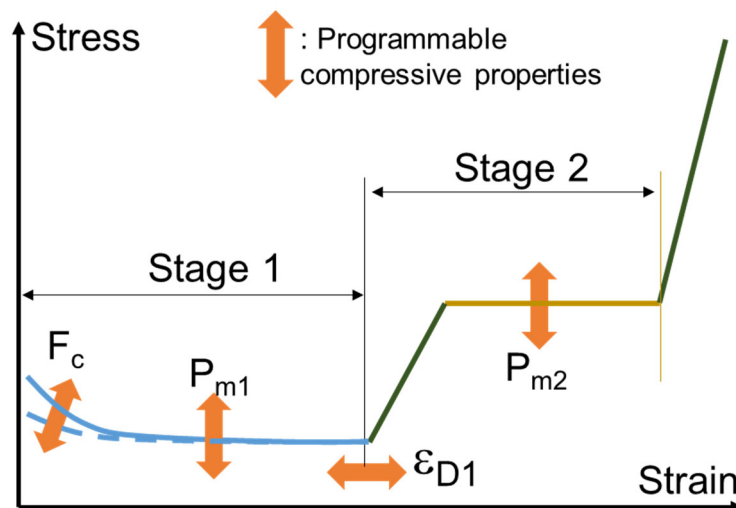
31 Cellular structures including lattice, foam, honeycomb and corrugated structures, often consist  
32 of unit cells formed by a group of the interconnected plate, sheets or struts [1]. Due to the  
33 advantages such as lightweight and high strength to weight ratio, these structures have been  
34 widely used and extensively studied [2-8]. The mechanical properties of these cellular  
35 structures can be architected based on the parameters such as geometries of the unit cell and  
36 the relative density. For instance, the microstructure of foam and lattice material can be  
37 categorized into bending or stretching dominated structures depending on the typical  
38 deformation of cell walls and struts, where stretching dominated structures often have  
39 significantly higher compressive strength than the bending dominated structures [1, 4]. The  
40 mechanical properties of honeycomb and corrugated structure are also governed by the  
41 geometries of the unit cell [9, 10].

42 Mechanical metamaterials are defined as a class of multiscale structures, which exhibit  
43 characteristics of unusual deformation or counterintuitive mechanical responses [11], such as  
44 negative Poisson's ratio [12, 13], negative thermal expansion [1], multi-stability [14-16],  
45 programmable stiffness [17]., etc. The unique mechanical properties of the structures are  
46 generated mostly from the structuring of the unit element rather than the mechanical behaviour  
47 of the base material. For example, by arranging the struts and cell wall within each unit cell of  
48 a cellular structure, auxetic metamaterials with a negative Poisson's ratio could be achieved in  
49 2D [18, 19] and 3D [20, 21], even though the base material possess a positive Poisson's ratio.  
50 Under compression, the auxetic material contracts in the direction perpendicular to the  
51 direction of compression, resulting in an enhanced energy absorption against impact loads [22-  
52 24].

53 Origami structures have received wide attention in the area of mechanical metamaterials, as  
54 sheet material can be transformed into complex geometrical structures through coordinated  
55 folding [17, 25, 26]. Geometries of origami mechanical metamaterial are governed by  
56 parameters such as the magnitude, quantity, sequence, location and direction of the folds [1].  
57 Kinematics and mechanical properties of origami structures, especially Miura-type origami  
58 structures, have been extensively studied [27-32]. Origami based mechanical metamaterials  
59 were investigated as well due to the unique properties, such as negative Poisson's ratio and  
60 self-locking. Stacked Miura-ori metamaterials were investigated for their folding kinematics  
61 and energy absorption capacity [25, 26, 33]. Origami metamaterial with graded stiffness was

62 proposed by varying the geometries on each stacked Miura-ori layer [34]. Origami-inspired  
63 deployable mechanical metamaterial with tunable stiffness was studied [35]. Miura-ori tubes  
64 assembled metamaterial with reconfigurable stiffness was proposed and investigated [36].  
65 Origami mechanical metamaterial with programmable two-stage stiffness via self-interlocking  
66 was studied [17]. Recently, the studies on Miura-ori based mechanical metamaterials with  
67 multiple stages of deformation were carried out [37, 38]. By combining the Miura-ori and  
68 honeycomb structure, a graded effect can be achieved through a developable creased pattern  
69 and good energy absorption capability has been demonstrated [37]. The programmability of  
70 Poisson's ratio and stiffness was investigated for multi-stage origami mechanical  
71 metamaterials based on curved-crease Miura-ori [38]. However, the crushing responses such  
72 as peak and mean crushing force of the multi-stage origami metamaterial have not been  
73 investigated.

74 In this study, a two-stage mechanical origami metamaterial with programmable crushing  
75 behaviour is proposed. The proposed metamaterial is developed by combining Miura-ori with  
76 rhombic honeycomb structure. Due to the different deformation of the Miura-ori and rhombic  
77 honeycomb sections, a two-stage crushing response can be achieved. The compressive  
78 properties, such as compressive strength and densification strain for both stages, can be  
79 programmed by adjusting the geometric parameters, as shown in Figure 1. Initial peak force  
80 during the first stage can be minimized and the commonly seen fluctuation of the crushing  
81 resistance on honeycomb structure is also minimized during the second stage deformation.

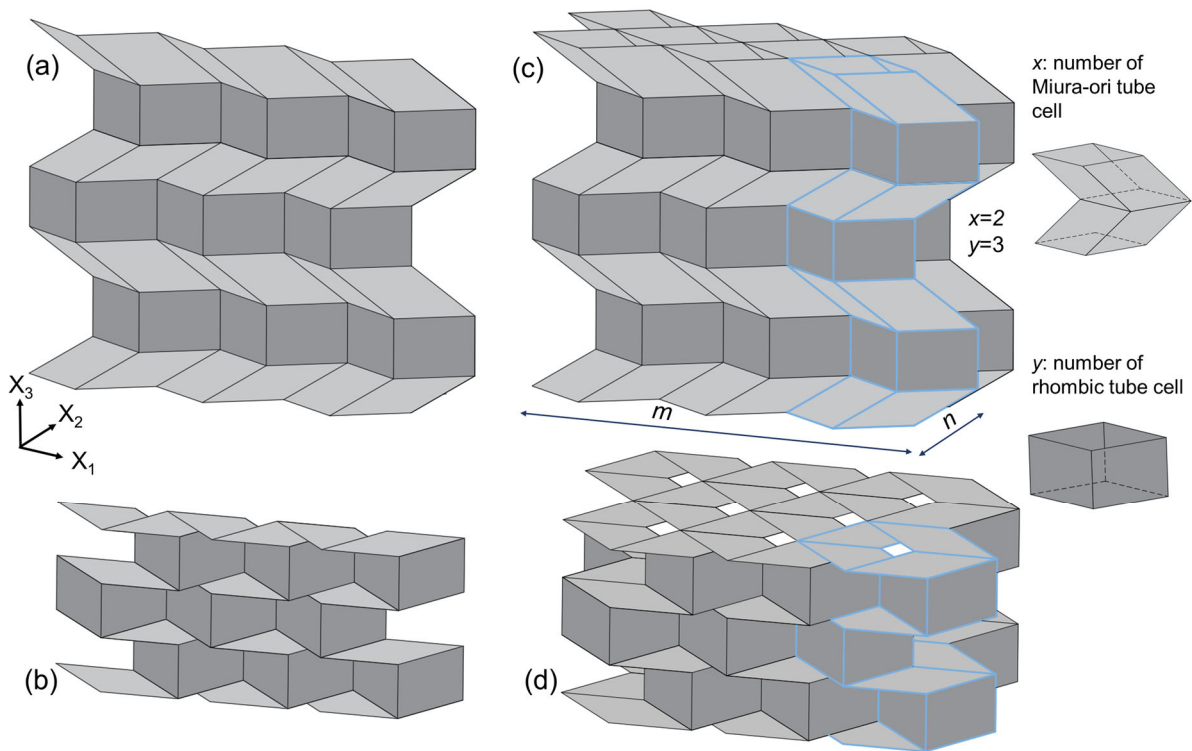


82

83 Figure 1. Illustration of the programmable compressive properties of the proposed origami  
84 metamaterial

85 The analytical model was firstly developed for the proposed structure to predict the mean  
 86 crushing force and determine the initial peak force during the first stage deformation under  
 87 quasi-static lateral crushing. Due to the equal and opposite Poisson's ratio in the two in-plane  
 88 directions, the stacked Miura-ori section expands and contracts in both in-plane directions  
 89 when subjected to the out-of-plane crushing, results in sliding of the edges at the contacting  
 90 surface. The additional crushing resistance contributed by friction during this sliding was  
 91 considered in the model. The analytical model was then verified against numerical simulations  
 92 and crushing tests. The programmability of initial peak force, mean crushing force and  
 93 densification strain against governing geometric parameters was analyzed using the validated  
 94 analytical model.

## 95 2 Geometric parameters



96

97 Figure 2. (a) Initial state and (b) end of the first stage deformation of single-sheet origami  
 98 metamaterial; (c) initial state and (d) end of the first stage deformation of origami metamaterial  
 99 with  $m=3$   $n=2$   $x=2$   $y=3$ ; unit column is marked out in blue lines

100 The proposed origami metamaterial combines the Miura-origami with rhombic honeycomb  
 101 structure. The Miura-type metamaterial is often referred to as the structure made of stacks of  
 102 Miura-ori sheet layers, which have a series of tessellated zig-zag crease patterns on each layer.  
 103 The stacked layers then form the Miura-type metamaterial with different compressive  
 104 properties in the different principle directions and a changing Poisson's ratio throughout

105 crushing [25]. The rhombic honeycomb structure has the profile of a parallelogram for each  
 106 unit cell instead of a hexagon on the conventional honeycomb structure. Similar to honeycomb  
 107 structure, rhombic honeycomb structure can provide high compressive resistance under out of  
 108 plane crushing [39]. The proposed origami metamaterial that combines Miura-type  
 109 metamaterial and rhombic honeycomb deforms in two stages under compression. The Miura-  
 110 ori portion of the structure deforms first, followed by the deformation on rhombic honeycomb  
 111 section.

112 The initial state and the end of the first stage deformation of the proposed structure are shown  
 113 in Figure 2. During the first stage of deformation, the faces of the Miura-ori sections undergo  
 114 bending deformation along the existing creases with minimal deformation on the faces,  
 115 therefore generate relatively low yet uniform crushing resistance throughout the deformation  
 116 [26]. Once the faces of Miura-ori sections flatten out, the layers of the honeycomb sections  
 117 start to buckle and provide a higher crushing resistance during the second deformation stage.

118 In this study, origami metamaterial with two layers of Miura-ori cell ( $x=2$ ) are divided and  
 119 separated by three layers of the rhombic honeycomb ( $y=3$ ) along  $X_3$  direction throughout this  
 120 paper. The metamaterial is divided into vertical “S-shaped” tubes with an array of  $m \times n$  unit  
 121 column in  $X_1$  and  $X_2$  directions, respectively. To analytically model the crushing response of  
 122 the proposed two-stage metamaterial, a unit column including  $x$  number of Miura-ori tube cell  
 123 and  $y$  number of rhombic tube cell is selected as marked out in blue lines in Figure 2. Geometric  
 124 parameters are shown in Figure 3. The vertices are the intersection of the edges and marked as  
 125  $A_{1-4}$ ,  $B_{1-4}$ ,  $C_{1-4}$ ,  $D_{1-4}$  in Figure 3 (b) along the top, middle and bottom sections of the Miura-ori  
 126 tube cell and rhombic tube cell. The pattern geometry is governed by six parameters only: the  
 127 edge length of the top section,  $a$ ,  $b$ ; the angle of the top face,  $\alpha$ ; the folding angle between a  
 128 top sidewall ( $A_1B_1B_2A_2$ ) and vertical plane ( $A_1B_1B_3A_3$ ),  $\theta$ ; the middle section height,  $h_2$ ; and  
 129 the sheet thickness,  $t$ . Other parameters can be determined as follows:

$$W = 2a \cdot \sin \theta \sin \alpha \quad (1)$$

$$h_1 = \frac{b \cdot \cos \theta \tan \alpha}{\sqrt{1 + \cos^2 \theta \tan^2 \alpha}} \quad (2)$$

$$L = 2a \sqrt{1 - \sin^2 \theta \sin^2 \alpha} \quad (3)$$

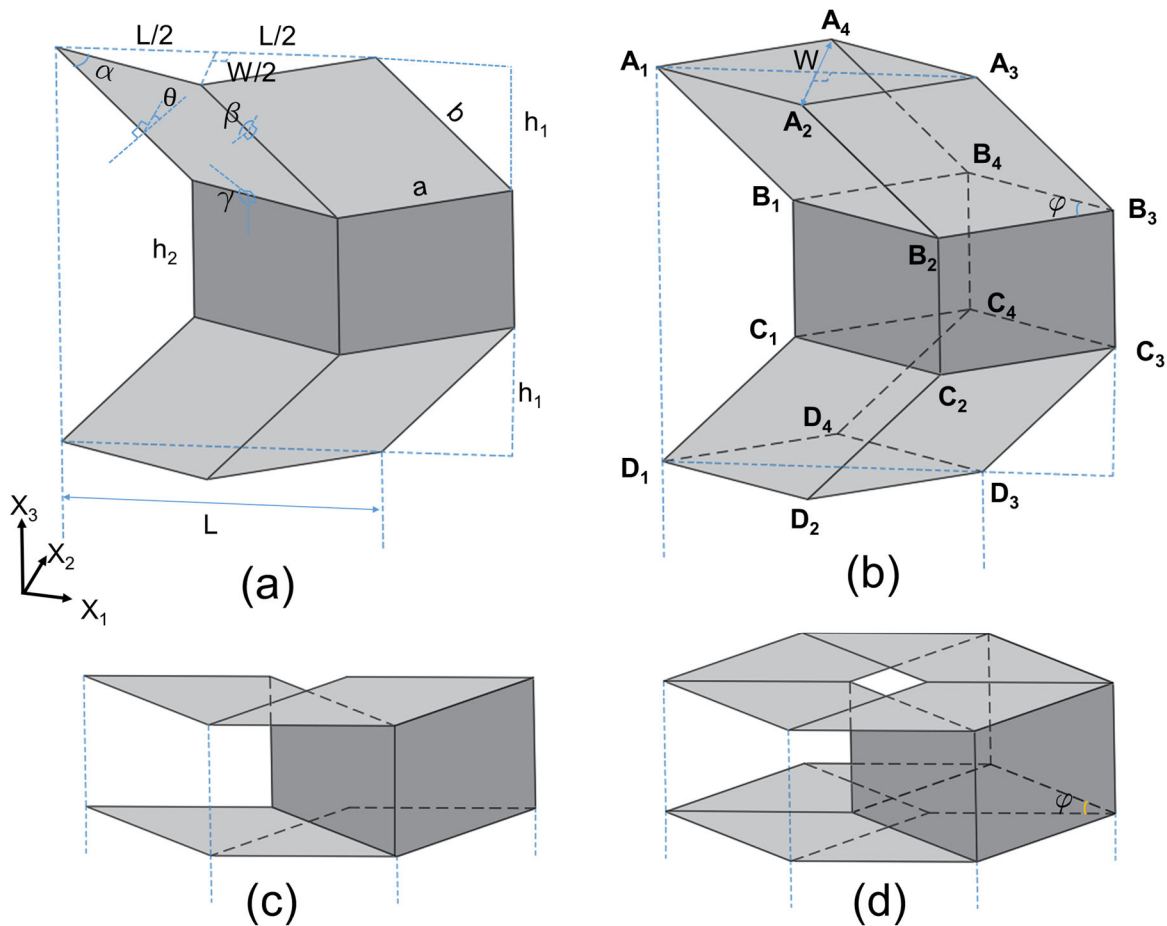
$$H = 2xh_1 + yh_2 \quad (4)$$

$$\sin\left(\frac{\varphi}{2}\right) = \sin\theta \sin\alpha \quad (5)$$

$$\sin\left(\gamma - \frac{\pi}{2}\right) = \frac{\cos\theta}{\cos\alpha \cdot \sqrt{1 + \cos^2\theta \tan^2\alpha}} \quad (6)$$

$$\rho_r = \frac{8xabs\sin\alpha \cdot t + 4yah_2t}{W \cdot L \cdot (2xh_1 + yh_2)} \quad (7)$$

130 where  $W$  and  $L$  are the width ( $A_2A_4$ ) and length ( $A_1A_3$ ) of the unit column,  $h_1$  is the height of  
 131 the half of a Miura cell;  $H$  is the overall height of a unit column,  $x$ ,  $y$  is the number of Miura-  
 132 ori cell and rhombic tube cell in a unit column,  $\varphi$  is the angle between the middle section faces  
 133 ( $B_2C_2C_3B_3$  and  $B_3C_3C_4B_4$ ), and  $\gamma$  is the angle between adjacent faces of the top and middle  
 134 sections ( $A_1B_1B_2A_2$  and  $B_1C_1C_2B_2$ ).  $\rho_r$  is the volumetric density.



135

136 Figure 3. Geometric parameters at the initial state for (a) sheet; (b) column cell; and at the end  
 137 of the first stage of deformation (c) sheet; (d) column cell

### 138 3 Analytical model

139 The analytical derivation of the crushing response of the proposed origami metamaterial can  
140 be divided into two parts owing to the two-stage deformation. Ductile metal sheet such as  
141 aluminium sheet is selected as the material for the proposed origami metamaterial, as it can  
142 undergo large plastic deformation and absorb energy. Furthermore, ductile metal sheet can be  
143 easily formed into desired shapes by press moulding. It is assumed that the strain hardening at  
144 the folding creases caused by press moulding fabrication is not considered and only plastic  
145 energy is considered during the compression of the proposed metamaterial.

#### 146 3.1 1<sup>st</sup> stage of deformation

147 In this section, the first stage of deformation is modelled. Throughout the 1<sup>st</sup> stage crushing,  
148 the bending deformation along the existing folding creases such as A<sub>1</sub>B<sub>1</sub>, B<sub>1</sub>B<sub>2</sub> and B<sub>1</sub>C<sub>1</sub> is  
149 expected with minimal buckling deformation on the faces, as suggested by the previous studies  
150 [26, 33]. Thus, only bending along the existing folding creases are considered during the first  
151 stage of deformation. Friction induced additional compression force is considered in this study,  
152 as the contacting base area is constantly changing during the 1<sup>st</sup> stage of crushing.

##### 153 3.1.1 Crushing force of each unit column

154 The crushing force of a unit column during the first stage deformation along X<sub>3</sub> direction,  $F$ ,  
155 can be expressed by including the bending in three groups of creases as follows

$$F \cdot x \cdot 2dh_1 = x \cdot 16bM_0d\theta + x \cdot 16aM_0d\gamma + y \cdot 4h_2M_0d\varphi \quad (8)$$

156 where  $x$ ,  $y$  is the number of Miura-ori unit cell and rhombic tube cell in each unit column and  
157  $M_0$  is the plastic bending moment per unit length [40] as expressed as

$$M_0 = \frac{\sigma_0 \cdot t^2}{4} \quad (9)$$

158 where  $\sigma_0$  and  $t$  are the yield strength and thickness of the metal sheet, respectively. Furthermore,  
159 the crushing force can be rewritten with respect to  $\theta$  based on equation (8):

$$F \cdot x \cdot 2 \frac{\partial h_1}{\partial \theta} d\theta = x \cdot 16bM_0d\theta + x \cdot 16aM_0 \frac{\partial \gamma}{\partial \theta} d\theta + y \cdot 4h_2M_0 \frac{\partial \varphi}{\partial \theta} d\theta \quad (10)$$

160 where



$$\frac{\partial h_1}{\partial \theta} = b \frac{\sin \theta \tan \alpha}{(1 + \cos^2 \theta \tan^2 \alpha)^{3/2}};$$

$$\frac{\partial \gamma}{\partial \theta} = \frac{\sin \theta \sec \alpha}{(1 + \cos^2 \theta \tan^2 \alpha)^{3/2} \sqrt{1 - \frac{\cos^2 \theta \sec^2 \alpha}{1 + \cos^2 \theta \tan^2 \alpha}}};$$

$$\frac{\partial \varphi}{\partial \theta} = \frac{2 \cos \theta \sin \alpha}{\sqrt{1 - \sin^2 \theta \sin^2 \alpha}}$$

161 are obtained from equation (1)- (6) and

$$F = 8M_0 \left[ \frac{(1 + \cos^2 \theta \tan^2 \alpha)^{3/2}}{\sin \theta \tan \alpha} + \frac{a \sqrt{1 + \cos^2 \theta \tan^2 \alpha}}{b \tan \alpha \sqrt{\cos^2 \alpha (1 + \cos^2 \theta \tan^2 \alpha) - \cos^2 \theta}} \right]$$

$$+ 4M_0 \frac{h_2}{b} \frac{y}{x} \frac{\cos \alpha (1 + \cos^2 \theta \tan^2 \alpha)^{3/2}}{\tan \theta \sqrt{1 - \sin^2 \theta \sin^2 \alpha}}$$

162 The critical crushing force or the initial peak force during the 1st stage of deformation,  $F_c$ ,  
 163 occurs at its initial position. Since only plastic deformation is considered,  $F_c$ , is therefore  
 164 obtained by substituting  $\theta = \theta_0$  into equation (12) and expressed as

$$F_c = 8M_0 \left[ \frac{(1 + \cos^2 \theta_0 \tan^2 \alpha)^{3/2}}{\sin \theta_0 \tan \alpha} + \frac{a \sqrt{1 + \cos^2 \theta_0 \tan^2 \alpha}}{b \tan \alpha \sqrt{\cos^2 \alpha (1 + \cos^2 \theta_0 \tan^2 \alpha) - \cos^2 \theta_0}} \right]$$

$$+ 4M_0 \frac{h_2}{b} \frac{y}{x} \frac{\cos \alpha (1 + \cos^2 \theta_0 \tan^2 \alpha)^{3/2}}{\tan \theta_0 \sqrt{1 - \sin^2 \theta_0 \sin^2 \alpha}}$$

165 where  $\theta_0$  is the initial angle of  $\theta$ .

166 The bending energy throughout the crushing is expressed from equation (8) as

$$E_b = 2x \int_{h_{1,0}}^{h_{1,f}} F \cdot dh_1 = x \int_{\gamma_0}^{\gamma_f} 16a \cdot M_0 \cdot d\gamma + x \int_{\theta_0}^{\theta_f} 16b \cdot M_0 \cdot d\theta + y \int_{\varphi_0}^{\varphi_f} 4h_2 M_0 d\varphi$$

$$= [16xa(\gamma_0 - \gamma_f) + 16xb(\theta_f - \theta_0) + 4yh_2(\varphi_f - \varphi_0)]M_0$$

167 The mean crushing force without friction,  $F_{m1}$ , is obtained by substituting equation (2):

$$F_{m1} = \frac{E_b}{2x(h_{1,0} - h_{1,f})} = \frac{[8xa(\gamma_0 - \gamma_f) + 8xb(\theta_f - \theta_0) + 2y(\varphi_f - \varphi_0)h_2]M_0}{xb \left[ \frac{\cos \theta_0 \cdot \tan \alpha}{(1 + \cos^2 \theta_0 \cdot \tan^2 \alpha)^{1/2}} - \frac{\cos \theta_f \cdot \tan \alpha}{(1 + \cos^2 \theta_f \cdot \tan^2 \alpha)^{1/2}} \right]}$$

168 where  $\gamma_0, \theta_0$  and  $\gamma_f, \theta_f$  are the initial and final angles of  $\gamma$  and  $\theta$  during the 1<sup>st</sup> stage  
 169 deformation. The after-crush angle of  $\theta_f$  can be calculated by assuming that the full  
 170 densification of Miura-ori section is reached at the end of stage 1 compression [26], the volume

171 consisting of eight faces on Miura-ori section becomes the same as the final volume of the  
 172 Miura-ori section in a unit column:

$$16 \cdot \frac{1}{2} ab \sin \alpha \cdot t = W_f \cdot L_f \cdot 2h_{1,f} \quad (16)$$

173 and the after-crush angle of  $\theta_f, \gamma_f$  are obtained as

$$\theta_f = \arccos \left( \sqrt{\frac{1}{2} - \sqrt{\frac{1}{4} - \frac{t^2}{a^2 \sin^2 \alpha}}} \right) \quad (17)$$

$$\gamma_f = \frac{\pi}{2} + \arcsin \frac{\cos \theta_f}{\cos \alpha \cdot \sqrt{1 + \cos^2 \theta_f \tan^2 \alpha}} \quad (18)$$

174 where both  $\theta_f, \gamma_f$  are close to but not equal to  $90^\circ$  due to column wall thickness.

175 The densification strain of the first stage of deformation,  $\varepsilon_{D1}$  is expressed as

$$\varepsilon_{D1} = \frac{2x(h_{1,0} - h_{1,f})}{2xh_{1,0}} = 1 - \frac{\cos \theta_f \sqrt{1 + \cos^2 \theta_0 \tan^2 \alpha}}{\cos \theta_0 \sqrt{1 + \cos^2 \theta_f \tan^2 \alpha}} \quad (19)$$

### 176 3.1.2 Mean crushing force of a structure with $m \times n$ unit columns

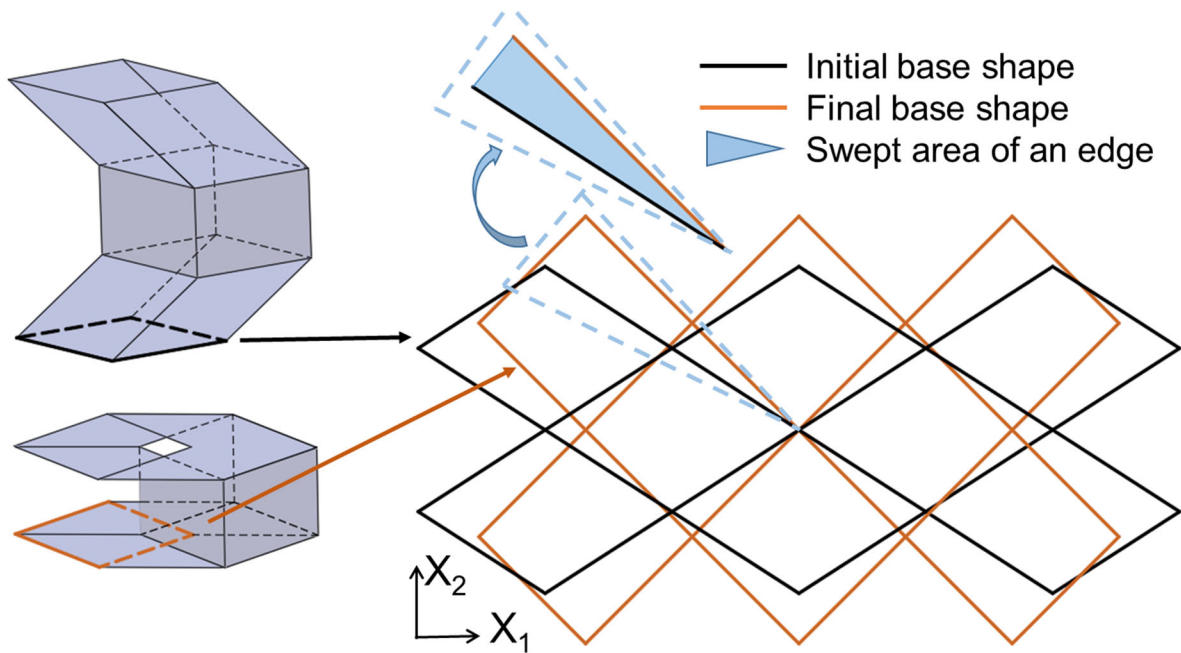
177 The crushing force with respect to angle  $\theta$  of a unit column, as well as its mean crushing force  
 178 are derived in the section 3.1.1. The mean crushing force of a structure with  $m \times n$  unit columns  
 179 is greater than the mean crushing force of the single unit column times  $m \times n$  because of the  
 180 strong bonding along the four longitudinal edges between the adjacent unit columns. It was  
 181 reported that the additional bending energy due to bonding between adjacent cells is equal to  
 182 the bending energy of the edge in the cell [6]. For example, the additional bending energy due  
 183 to bonding connection with the adjacent unit cells along  $A_1B_1$  is  $2bM_0d\theta$ , equals to the  
 184 bending energy of the edge  $A_1B_1$  itself. Therefore, the overall crushing force of an  $m \times n$  origami  
 185 metamaterial,  $F_{m1(m \times n)}$ , can be written as:

$$F_{m1(m \times n)} = \frac{mn \cdot E_b}{2x(h_{1,0} - h_{1,f})} + \frac{[(m-1)n + (n-1)m] \cdot [2xb(2\theta_f - 2\theta_0) + y(\varphi_f - \varphi_0)h_2] M_0}{2x(h_{1,0} - h_{1,f})} \quad (20)$$

186 where the first term is related to the  $m \times n$  individual unit columns and the second term is the  
 187 number of bonding connections  $(m-1)n + (n-1)m$ , multiplied by the bending energy along  
 188 each longitudinal bonding connection. Bending energy during the first stage deformation of a  
 189 unit column,  $E_b$ , is given in equation (14).

190 3.1.3 Additional crushing resistance contributed by friction

191 The additional crushing resistance contributed by friction is considered in this section. During  
 192 the first stage of compression, contacting edges of the structure on both top and bottom  
 193 interfaces move along the two in-plane directions. Due to friction between the pressing surfaces  
 194 and the top/ bottom edges, this sliding movement of the contacting top and bottom edges results  
 195 in additional work done which is proportional to the swept area of the edges on the contacting  
 196 surfaces throughout the first stage of deformation. An example of a structure with 3×2 unit  
 197 columns is shown in Figure 4, under lateral compression, where the proposed origami  
 198 metamaterial expands in  $X_2$  direction and contracts in  $X_1$  direction.



199

200 Figure 4. Illustration of the initial, final base shape and the swept area of an edge at the  
 201 contacting surface on a structure with 3×2 unit columns

202 The work done due to the friction can be expressed as

$$E_f = \frac{2F_f}{4a \cdot mn} \cdot A_{sweep} = \frac{\mu F_{m1(m \times n)}}{2amn} \cdot A_{sweep} \quad (21)$$

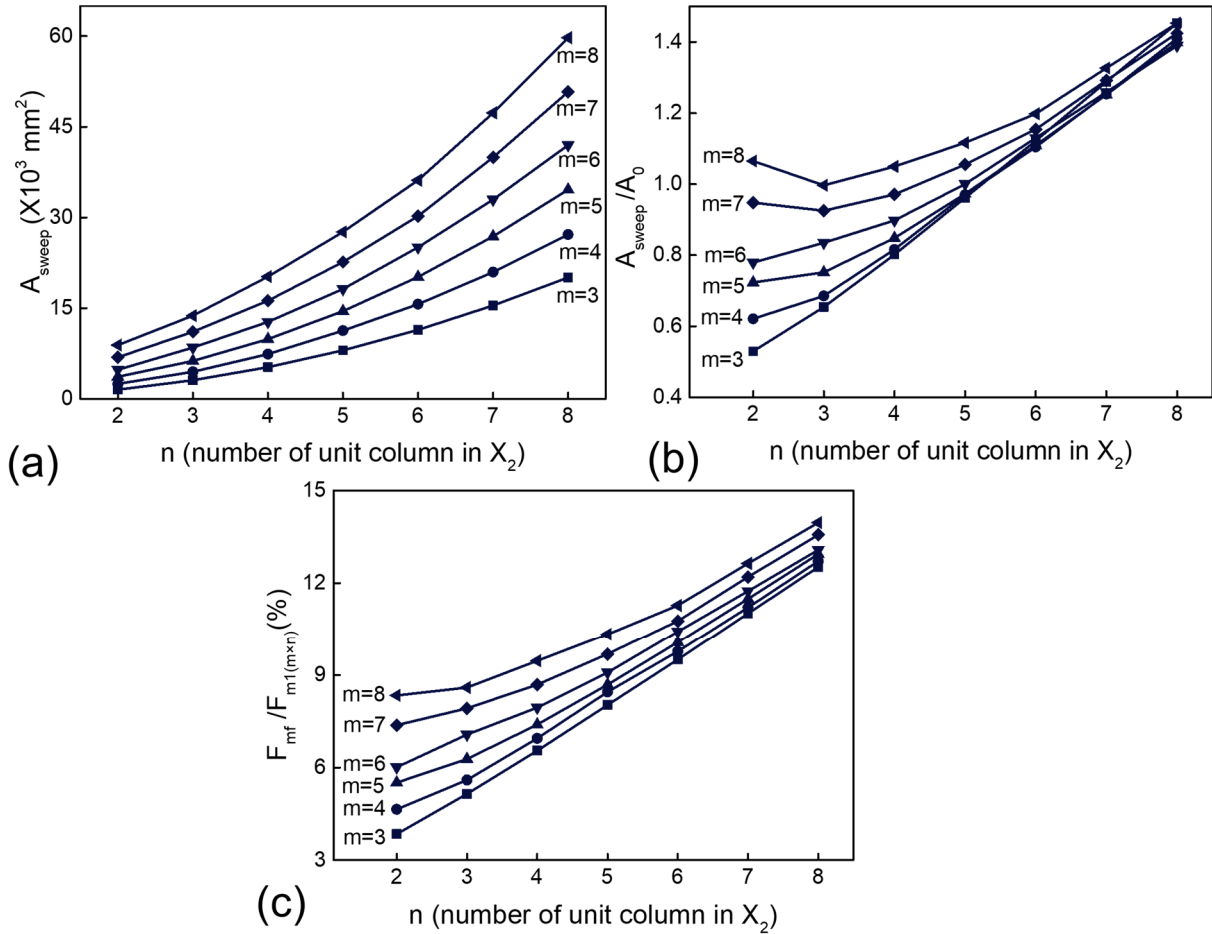
203 where  $F_f$  is the average friction force on contacting edges of the whole structure during the  
 204 first stage of deformation,  $\mu$  is the friction coefficient, and  $A_{sweep}$  is the swept area of edges on  
 205 contacting surfaces during the first stage of deformation,  $m$ ,  $n$  are the number of unit column  
 206 along the two in-plane directions. The friction force is assumed to be evenly distributed on the  
 207 contacting edges over the total length of  $4a \cdot mn$ . It should be noted that a generic formula for  
 208 swept area,  $A_{sweep}$ , of any given  $m \times n$  structure is difficult to derive, and the swept area is

209 measured in AutoCAD in this study. The compression force on the whole structure caused by  
 210 friction,  $F_{mf}$ , is obtained by using work done by friction during the first stage of deformation  
 211 divided by the compressed height as

$$F_{mf} = \frac{E_f}{2x(h_{1,0} - h_{1,f})} = \frac{\mu F_{m1(m \times n)} \cdot A_{sweep}}{4amn x(h_{1,0} - h_{1,f})} \quad (22)$$

212 and the contributing factor of friction is derived as

$$\frac{F_{mf}}{F_{m1(m \times n)}} = \frac{\mu \cdot A_{sweep}}{4amn \cdot x(h_{1,0} - h_{1,f})} \quad (23)$$



213

214 Figure 5. (a) Total swept area of interface edges after the first stage of crushing ( $A_{sweep}$ ); (b)  
 215 ratio of swept area ( $A_{sweep}$ ) to initial base area ( $A_0$ ); (c) ratio of friction contributed crushing  
 216 force ( $F_{mf}$ ) to the mean crushing force without considering friction ( $F_{m1(m \times n)}$ ) during the first  
 217 stage of deformation; with respect to the different number of the unit column ( $m, n$ );  $a=b=20$   
 218 mm,  $x=2$ ,  $y=3$ ,  $h_l=10.83$  mm,  $\alpha=45^\circ$ ,  $\theta_0=50^\circ$ ,  $\mu=0.25$

219 The contribution factor increases with the number of unit column  $m \times n$ , due to the non-linear  
 220 increase of bottom edge swept area with respect to the unit column number. Example of the  
 221 influence of friction for the proposed structure with different array configurations is shown in

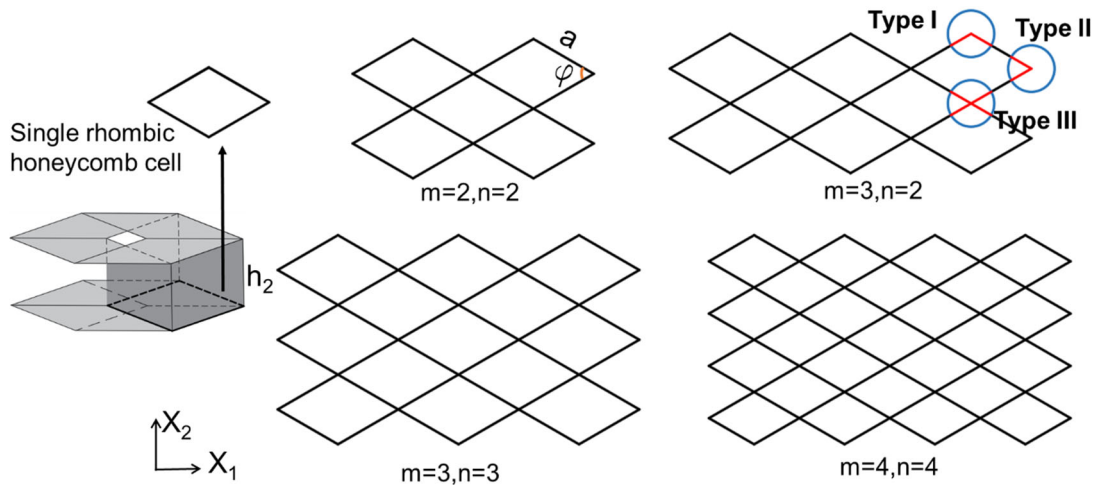
222 Figure 5. For the given pattern geometry, the friction contribution factor of mean crushing force  
 223 increases from around 4% to 14%, when the number of unit column of the metamaterial  
 224 increases from  $3 \times 2$  to  $8 \times 8$ , because of the non-linear increase of the swept area at contacting  
 225 surfaces.

226 The mean crushing force of the structure during the first deformation stage with friction  
 227 contribution ( $F_{mf}$ ) considered can be expressed as

$$P_{m1} = F_{m1(m \times n)} + F_{mf} = F_{m1(m \times n)} \left[ 1 + \frac{\mu \cdot A_{sweep}}{4amn x(h_{1,0} - h_{1,f})} \right] \quad (24)$$

### 228 3.2 2<sup>nd</sup> stage of deformation

229 At the end of the first stage deformation, the Miura-ori section of the structure becomes fully  
 230 crushed with a residual height of  $2xh_{1,f}$  and this section is assumed to be densified. During the  
 231 second stage of deformation, the buckling deformation of the faces of the rhombic honeycomb  
 232 becomes dominate, resulting in a significantly higher crushing resistance than the first stage of  
 233 deformation. Grid structures, including square and circular hollow columns [40-43], multi-cell  
 234 columns [44-47], square and rhombic honeycomb have been extensively studied [7, 10, 39].  
 235 Analytical models with different coefficients of the crushing resistance of these structures were  
 236 developed and compared. Therefore, an established analytical model for rhombic honeycomb  
 237 is used for this study where inextensional damage mode was selected for corner element with  
 238 an angle less than  $135^\circ$  [39, 45]. Three energy dissipation mechanisms were considered in this  
 239 model, including the bending of the stationary hinge lines, the rolling of the moving hinge lines  
 240 as well as the membrane deformation in the forming of the toroidal surface [6, 39].



241

242 Figure 6. Rhombic honeycomb layer of origami metamaterial with different number of unit  
 243 columns ( $m \times n$ )

244 The mean crushing force of rhombic honeycomb layer is calculated by summing up the mean  
 245 crushing force of the two types of the corner elements and the ‘X-shape’ intersection elements,  
 246 as shown in Figure 6. The mean crushing force for each Type I corner element,  $F_{m2,I}$ , [39] can  
 247 be expressed as

$$F_{m2,I} = \frac{1}{2\varepsilon_{D2}} \sigma_0 \left(\frac{a}{2}\right)^{\frac{1}{3}} t^{\frac{5}{3}} \sqrt{\frac{2\pi \tan(\pi/2 - \varphi/2)}{0.163 \left[ \tan(\pi/2 - \varphi/2) + 0.06 / \tan(\pi/2 - \varphi/2) \right]}} \quad (25)$$

248 and the mean crushing force for each Type II corner element,  $F_{m2,II}$ , [39] is expressed as

$$F_{m2,II} = \frac{1}{2\varepsilon_{D2}} \sigma_0 \left(\frac{a}{2}\right)^{\frac{1}{3}} t^{\frac{5}{3}} \sqrt{\frac{2\pi \tan(\varphi/2)}{0.163 \left[ \tan(\varphi/2) + 0.06 / \tan(\varphi/2) \right]}} \quad (26)$$

249 The mean crushing force for each Type III ‘X-shape’ intersection element,  $F_{m2,III}$ , [39] is  
 250 predicted by

$$F_{m2,III} = \frac{2}{\varepsilon_{D2}} \sigma_0 \left(\frac{a}{2}\right)^{\frac{1}{2}} t^{\frac{3}{2}} \sqrt{\pi \tan(\varphi/2) + 4\pi \sec(\varphi/2) \left(\frac{2t}{b}\right)^{\frac{1}{2}}}; \text{ for } \varphi \in (0, \pi/2]$$

$$F_{m2,III} = \frac{2}{\varepsilon_{D2}} \sigma_0 \left(\frac{a}{2}\right)^{\frac{1}{2}} t^{\frac{3}{2}} \sqrt{\pi \tan(\pi/2 - \varphi/2) + 4\pi \sec(\pi/2 - \varphi/2) \left(\frac{2t}{b}\right)^{\frac{1}{2}}}; \text{ for } \varphi \in [\pi/2, \pi) \quad (27)$$

251 where  $a$  is the edge length of the cell,  $t$  is the wall thickness,  $\sigma_0$  is the yield strength of the base  
 252 material,  $\varphi$  is the angle of the adjacent cell faces. It should be noted that the angle  $\varphi$  changes  
 253 during the first stage of deformation, but remains unchanged during the second stage of  
 254 deformation.  $\varphi$  used in this section can be obtained by substituting the final angle,  $\theta_f$ ,  
 255 obtained from equation (17), into equation (5).

256 Densification strain of the second stage deformation,  $\varepsilon_{D2}$ , or the effective crushing coefficient  
 257 referred in the previous studies [6, 39, 40], is the ratio of crushed distance and the original  
 258 height of the rhombic honeycomb section. It was recorded to be 73% for a square tube under  
 259 the inextensional mode of deformation [41], it can be also estimated from numerical results  
 260 [39]. The rhombic honeycomb sections in the proposed structure are divided by the Miura-ori  
 261 sections into multiple layers, resulting in a lower ratio of cell edge length to height than that of  
 262 square tubes. This could lead to a lower effective crushing coefficient due to uncompleted  
 263 folding after crushing. Thus, in this study, the effective crushing coefficient ( $\varepsilon_{D2}$ ) is obtained

264 from the numerical simulation rather than 73% of the conventional square tubes reported in  
 265 [41].

266 The mean crushing force of origami metamaterial with  $3 \times 2$  unit columns during the second  
 267 stage  $P_{m2}$  is

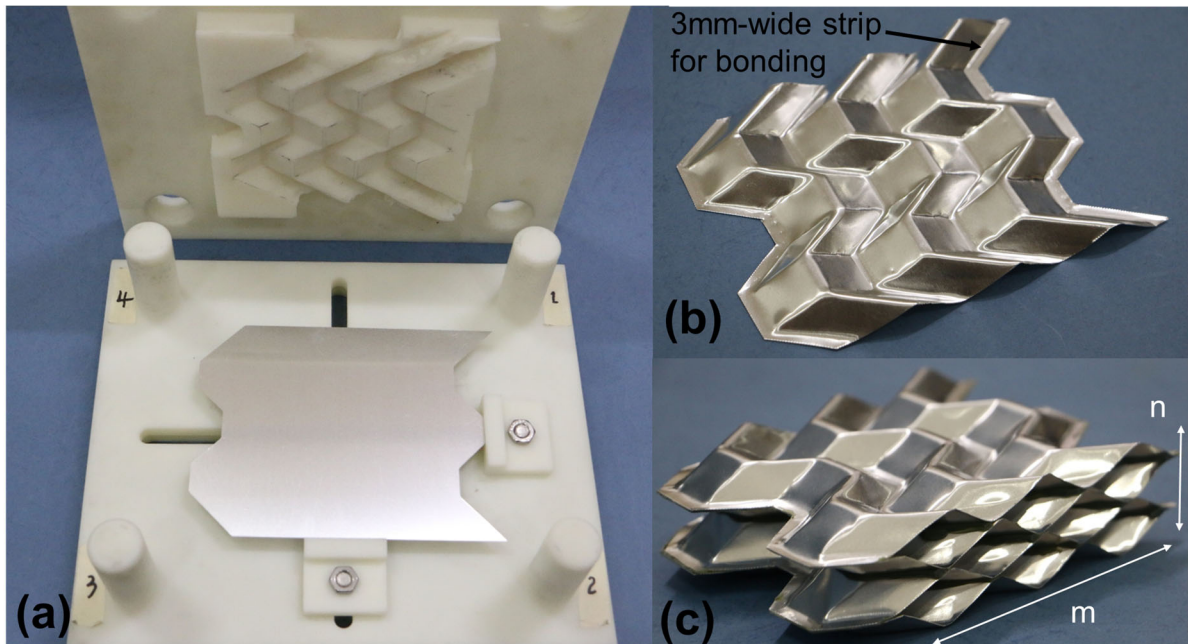
$$P_{m2} = 2mF_{m2,I} + 2nF_{m2,II} + [(m-1)n + (n-1)m]F_{m2,III} \quad (28)$$

268 To summarize, the mean force of an  $m \times n$  metamaterial without considering friction at  
 269 interfaces can be calculated by using equation (20) for the displacement in  $X_3 \in (0, \varepsilon_{D1} 2xh_{1,0}]$   
 270 and equation (28) for the displacement in  $X_3 \in (\varepsilon_{D1} 2xh_{1,0}, \varepsilon_{D1} 2xh_{1,0} + \varepsilon_{D2} yh_2]$ . Equation (24)  
 271 can be used to calculate the 1<sup>st</sup> stage mean crushing force considering friction, where swept  
 272 area,  $A_{sweep}$  needs to be measured for a specific  $m \times n$  metamaterial. The overall densification  
 273 strain can be expressed as

$$\varepsilon_D = \frac{\varepsilon_{D1} 2xh_{1,0} + \varepsilon_{D2} yh_2}{2xh_{1,0} + yh_2} \quad (29)$$

## 274 4 Model verification

### 275 4.1 Quasi-static crushing test



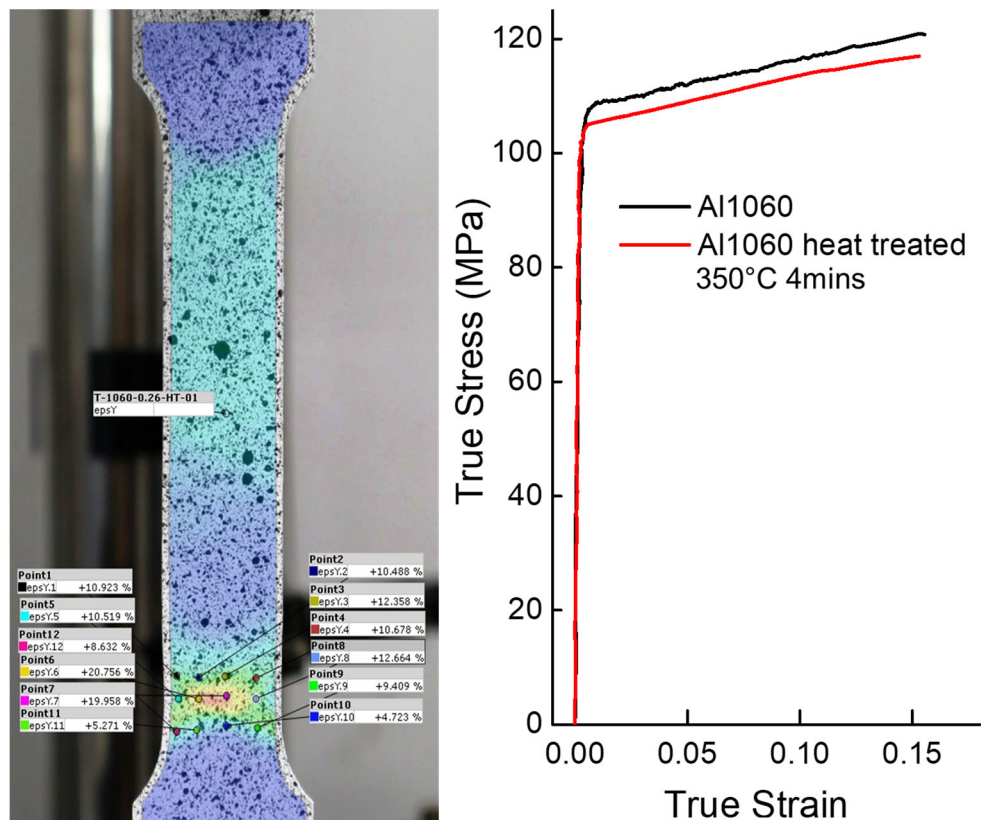
276 Figure 7. (a) 3D-printed pressing mould for layer fabrication; (b) pressed layer after trimming  
 277 along the outer edges; (c)  $3 \times 2$  specimen by bonding four heat-treated layers

279 Quasi-static crushing tests of  $3 \times 2$  ( $m \times n$ ) origami metamaterials with and without heat treatment  
 280 were carried out. The geometry parameters used for this metamaterials are given in Table 1.

281 To form the specimen with  $3 \times 2$  unit columns, each layer was firstly fabricated by mould-  
 282 pressing from Al 1060 metal sheet with a thickness of 0.26 mm, as shown in Figure 7 (a). The  
 283 one-stage pressing mould was 3D printed using photopolymer. The pressed sheets were  
 284 trimmed with a 3 mm-wide strip left on each end of the sheet for better gluing as shown in  
 285 Figure 7 (b). Heat-treatment at  $350^\circ\text{C}$  for 4 minutes was applied to remove the residual stress  
 286 caused in the mould pressing fabrication along the crease lines. As shown in Figure 7 (c), four  
 287 heat-treated layers were then bonded together by using Ergo 1665 NB adhesive with tensile  
 288 strength over 22 MPa and curing time of at least 24 hrs. SUNS® UT4304 universal testing  
 289 machine was used for the quasi-static compression tests. The metamaterial specimens were  
 290 placed between the cross-head and supporting disc without the constraint at interfaces. Both  
 291 the supporting disc and cross-head have 150 mm diameter. The specimens were then crushed  
 292 under quasi-static loading condition with a constant speed of 2 mm/min.

a (mm)	b (mm)	t (mm)	h <sub>1</sub> (mm)	h <sub>2</sub> (mm)	H (mm)	x	y	$\alpha$	$\theta_0$
20	20	0.26	10.8	15	88.3	2	3	$45^\circ$	$50^\circ$

293 Table 1. Geometric parameters for the metamaterial used in quasi-static tests



294

295 Figure 8. True tensile stress-strain curves of Al 1060 aluminium sheet with and without heat  
 296 treatment, where the strain was measured using the 2D-DIC technique



297 The tensile tests were also carried out for the heat-treated and unheat-treated Al 1060 sheet  
298 used in the specimen preparation. The quasi-static loading rate was kept constant at 0.7  
299 mm/min. The engineering stress-strain curves were firstly captured using 2-dimensional direct  
300 image correlation (DIC) technique and converted into true stress-strain curves, as shown in  
301 Figure 8. A slight reduction around 3 MPa was observed while Young's modulus remained the  
302 same. This reduction is caused by releasing a portion of the residual stress during heat treatment,  
303 as the sheet could be cold-rolled in the fabrication stage with work hardening. The density of  
304 Al 1060 is  $2710 \text{ kg/m}^3$ , and yield strength of Al 1060 is 110 MPa and 106 MPa, before and  
305 after  $350^\circ\text{C}$  heat treatment, respectively.

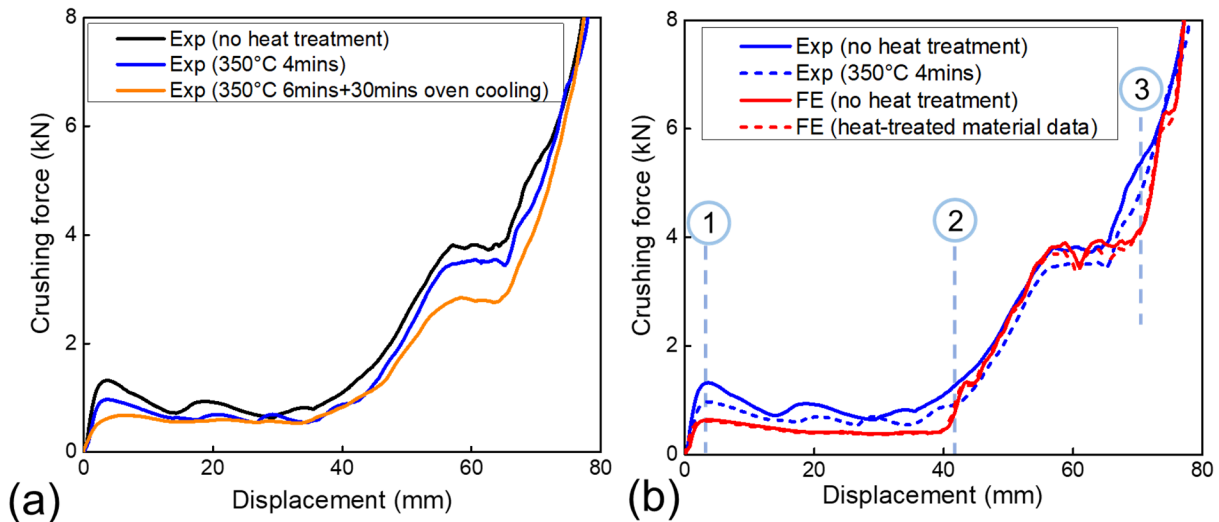
## 306 4.2 Finite element analysis

307 Numerical analysis of the tested specimens was conducted using FE software LS-DYNA.  
308 Piecewise linear plastic model was used for modelling the aluminium sheets of the  $3 \times 2$  (m $\times$ n)  
309 origami metamaterials. The material properties measured including true stress-strain data for  
310 both cases were implemented in the material model. For simplicity, the bonding connections  
311 between the layers of the specimen were not included. Instead, the layers of pressed sheets  
312 were modelled as a whole structure. Mesh size of 1 mm was selected after the convergence  
313 tests. Both cross-head and support plate were modelled as rigid blocks. The friction between  
314 interfaces and within the structure was considered with a coefficient of 0.25 [48]. The crushing  
315 speed was set to be constant as 1m/s, which was found to be sufficient to simulate the quasi-  
316 static loading condition with the kinetic energy to internal energy less than 5% [48, 49].

## 317 4.3 Model validation

318 Load-displacement curves from experiments and FE analysis for different heat-treated  
319 conditions are firstly compared, as shown in Figure 9. The folding process induces residual  
320 stress via work hardening as the aluminium sheets are deformed plastically and lead to a higher  
321 material strength near the creases. This folding induced residual stress at the creases results in  
322 a higher crushing resistance of the 1st stage of deformation where most deformation occurs  
323 near the creases. Thus, higher crushing resistance during the 1<sup>st</sup> stage of deformation is  
324 observed in the test without heat treatment as compared to FE result, as marked out in blue and  
325 red solid lines in Figure 9 (b). The heat treatment was then carried out aiming to release the  
326 residual stress along the creases. The crushing resistance, especially the initial peak force, is  
327 reduced with 4 minutes of annealing, but the discrepancies can still be noticed during the 1<sup>st</sup>  
328 stage crushing as marked in blue and red dash lines, indicating residual stress near creases from

329 mould-pressing is not fully released. However, further annealing leads to a noticeable drop in  
 330 crushing resistance in the 2<sup>nd</sup> stage. This is because the residual stress from metal sheet  
 331 fabrication is released during the longer annealing process, besides of that from mould-pressing  
 332 as intended. Therefore, a 4-minute heat treatment is used for the study despite the mould-  
 333 pressing induced residual stress is not fully released. In other word, the crushing resistance  
 334 during the 1<sup>st</sup> stage of deformation from the test is expected to be higher than that of the  
 335 analytical and FE results.

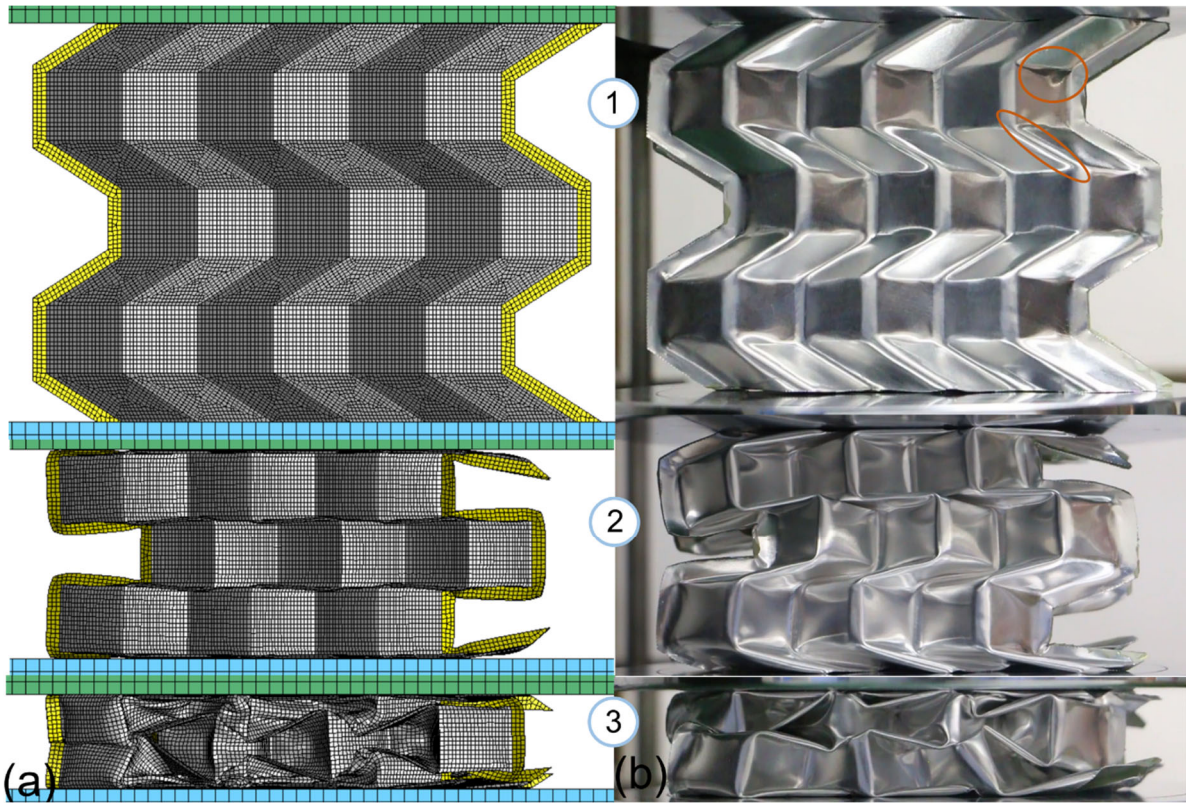


336

337 Figure 9. (a) Load-displacement curves from quasi-static crushing tests; (b) comparisons  
 338 between FE and test data; with the initial state, end of the first and second stage deformation  
 339 marked out as (1)/(2)/(3)

340 Three key states, i.e. initial, end of the 1<sup>st</sup> and 2<sup>nd</sup> stage of deformation from tests and FE results,  
 341 corresponding to the three turning points in load-displacement curves (Figure 9 b) are  
 342 compared in Figure 10. The deformation modes are matched between test and FE results,  
 343 although the deformation mode in FE seems more ideal with minimal face deformation during  
 344 the 1<sup>st</sup> stage of deformation. It can be observed that the crease lines of the prepared specimens  
 345 were not perfectly pressed with a noticeable bending radius around the designed lines, and  
 346 slight imperfection was induced on the faces during the mould-pressing process. This imperfect  
 347 folding leads to the slightly more face deformation in testing during the 1<sup>st</sup> stage of deformation  
 348 while only bending deformation along crease lines are observed from FE results. The face  
 349 deformation also contributes to the slightly higher crushing resistance in tests than FE results.  
 350 At the initial state of the 2<sup>nd</sup> stage of deformation, the offsets between rhombic layers leads to  
 351 a local buckling at the contacted areas rather than the multiple folding deformations common  
 352 in honeycomb and tubular structure. Thus, no initial peak is observed during the 2<sup>nd</sup> stage of

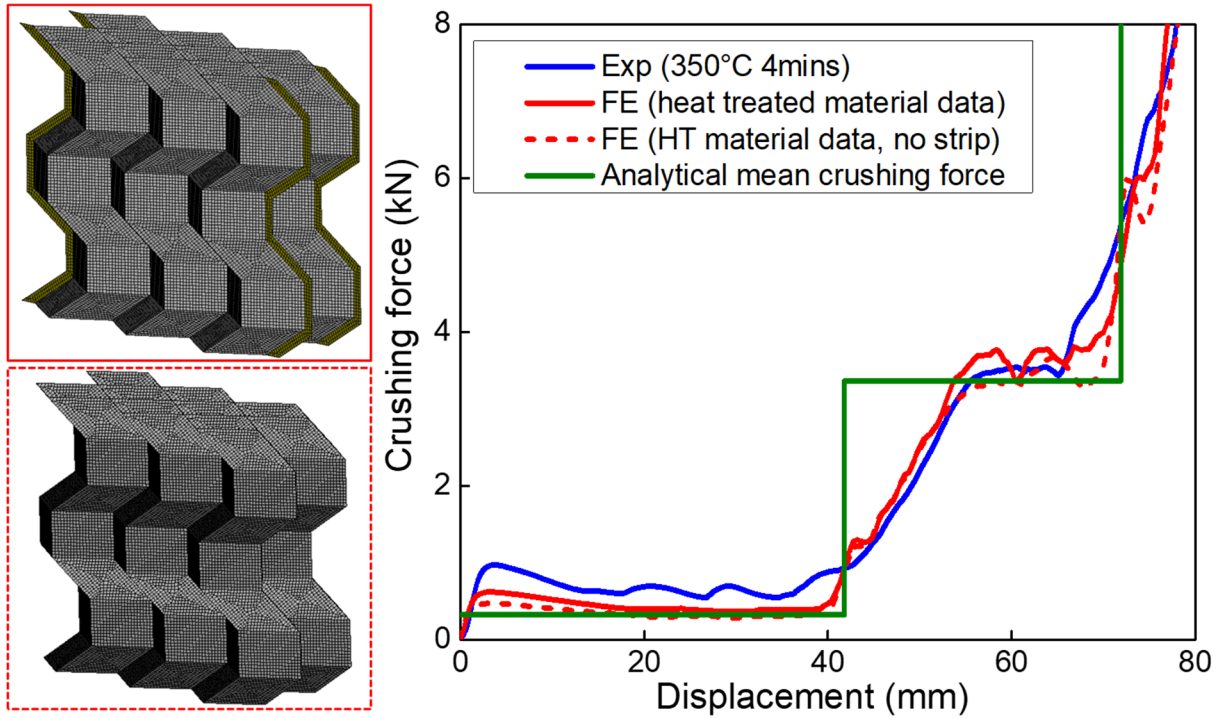
353 deformation and it takes longer distance for the layer to reach plateau resistance. The offsets  
 354 between rhombic honeycomb layers are governed by the geometric parameters of the Miura  
 355 section, and there could be infinite number of possible values, where each offset between layers  
 356 could affect the initial stage of deformation. Therefore, the limitation in analytical model at the  
 357 initial stage during the deformation of rhombic honeycomb layer is noted.



358 (a) (b)  
 359 Figure 10. Deformation modes comparison between (a) FE result and (b) quasi-static test, at  
 360 the initial stage (1), end of the first (2) and end of the second stage of deformation (3)

361 To better validate the analytical model, a numerical model without the strips which were used  
 362 for bonding in the tests was then constructed based on the previously verified FE model, as  
 363 shown in Figure 11. The analytical model for predicting the mean crushing forces of both stages  
 364 of the metamaterial is compared with the numerical model without the bonding strips. The FE  
 365 model without the strips shows a slightly lower crushing resistance and it is well matched with  
 366 the analytical model of the mean crushing force prediction. The mean crushing forces of  
 367 numerical results were taken average between initial, densification of the 1<sup>st</sup> and 2<sup>nd</sup> stage of  
 368 deformation, as listed in Table 2. The mean crushing force  $P_{m1}$ , and densification strain,  $\varepsilon_{D1}$ ,  
 369 of the 1<sup>st</sup> stage of deformation are in good agreement with errors less than 7.5%, while the  
 370 analytical mean crushing force of the 2<sup>nd</sup> stage,  $P_{m2}$ , is overestimated as compared to FE model.  
 371 The overestimation is caused by the change of deformation mode at the initial stage of the 2<sup>nd</sup>

372 deformation stage. The analytical model of the 2<sup>nd</sup> stage mean crushing force is based on the  
 373 classic kinematic model of the square tubes where the multiple folds are presented on the  
 374 structures. For this proposed metamaterial, the rhombic honeycomb layers may not be stacked  
 375 perfectly depending on the geometries and could lead to localized buckling deformation near  
 376 the contacts at the beginning as shown in Figure 10. However, the overall trend and the mean  
 377 crushing force at both deformation stages are in good agreement between FE and analytical  
 378 model.



379  
 380 Figure 11. Load-displacement response comparison among test, FE and analytical model; Note:  
 381 two FE models (with and without the 3mm strip) marked as solid and red dash lines are  
 382 included to respectively represent the test and analytical model

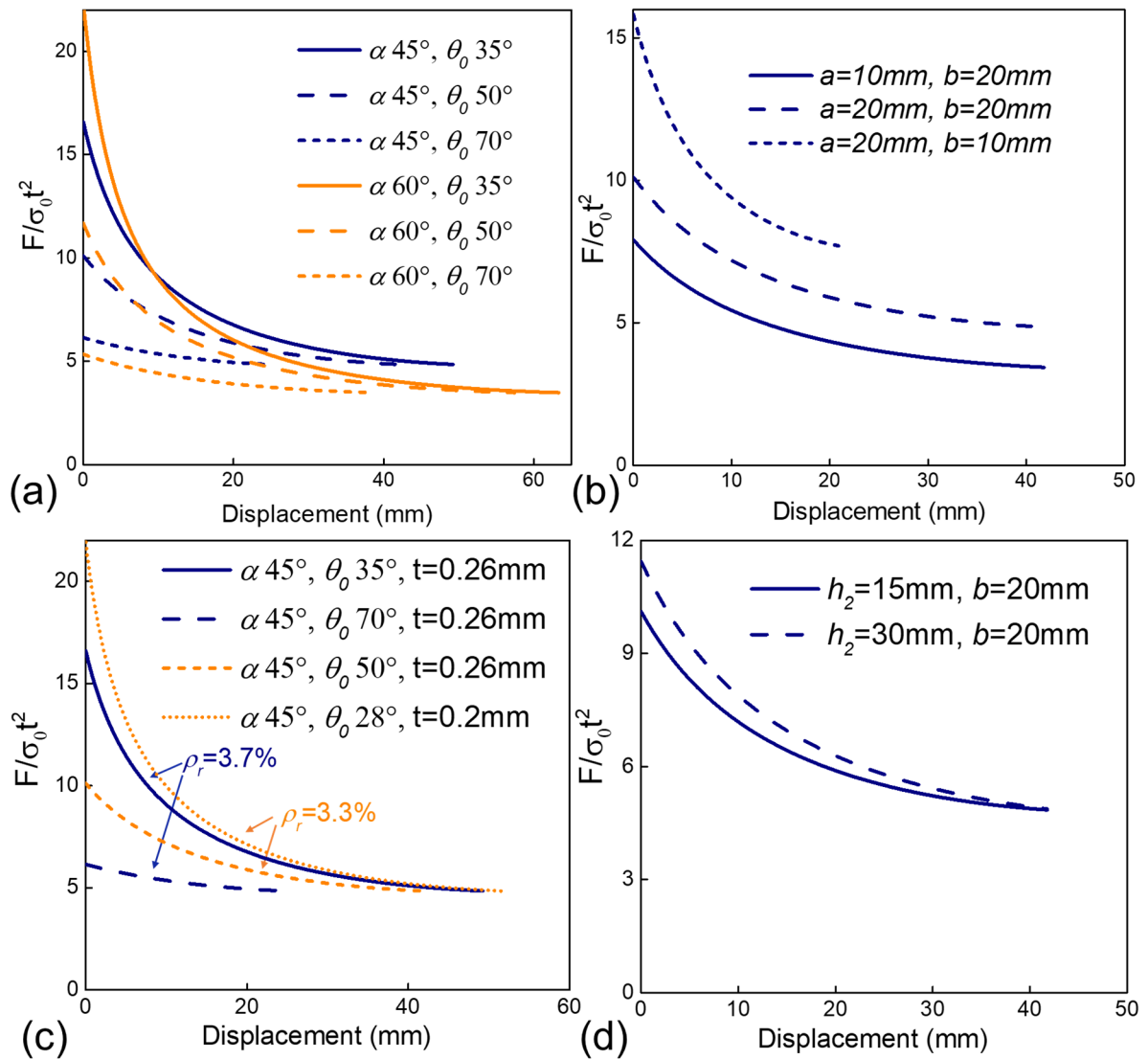
$P_{m1}$ (kN)			$\epsilon_{DI}$			$P_{m2}$ (kN)		
FE	Analytical	Error	FE	Analytical	Error	FE	Analytical	Error
0.347	0.321	-7.5%	0.933	0.965	3.4%	2.775	3.364	21.2%

383 Table 2. Mean crushing force comparison between analytical and numerical model without  
 384 strip

## 385 5 Programmable compressive properties

### 386 5.1 Initial peak force

387 As an important criterion for energy absorbing structures, initial peak force must be kept below  
 388 the threshold that would lead to damage or injury to the protected personnel and structures [50].  
 389 For common cellular structures such as honeycomb [10], lattice [51] and foam [52], both the  
 390 initial peak force and mean crushing resistance are in power law relationship with the relative  
 391 density. Therefore, the structure with a higher energy absorption capacity often has a high  
 392 initial peak force. For the proposed origami metamaterial, the initial peak force could be  
 393 significantly reduced with minimal reduction in mean crushing resistance by adjusting the  
 394 related geometric parameters.



395

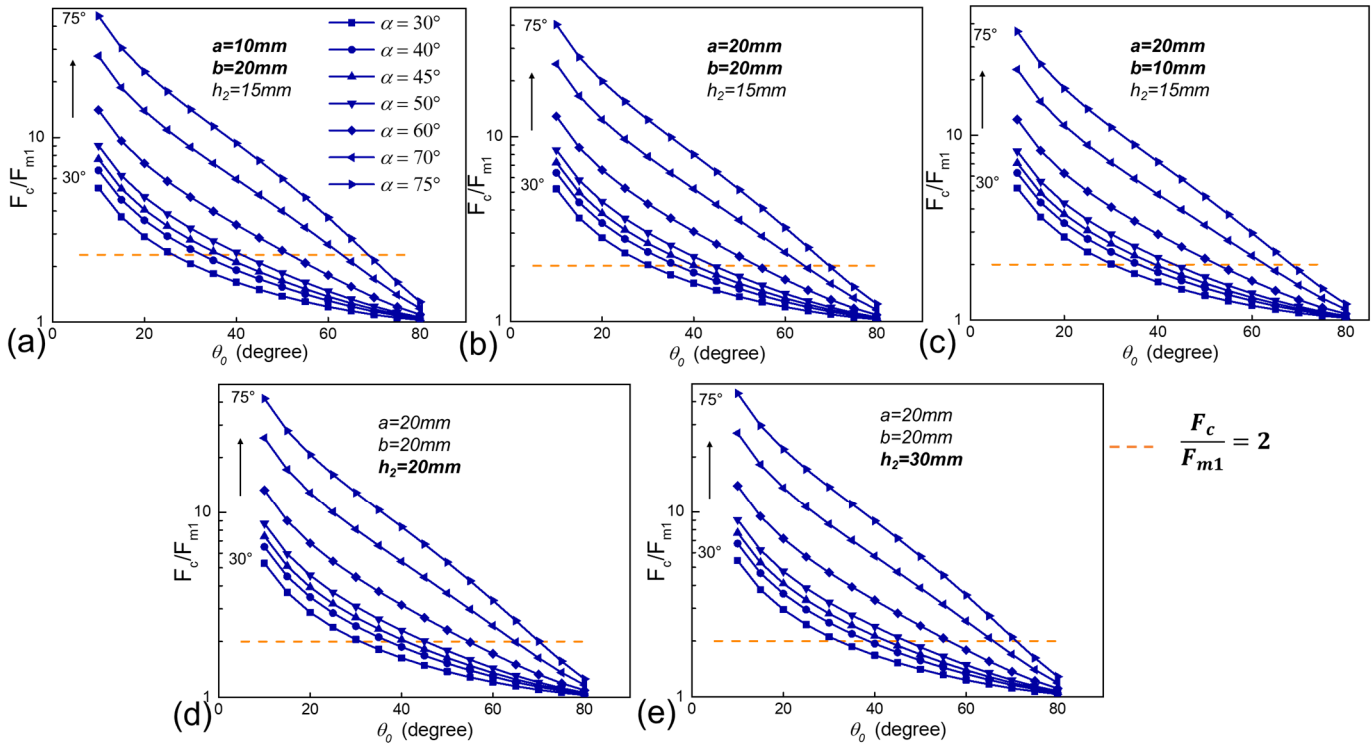
396 Figure 12. Comparison of normalized force-displacement curves of a unit column with  
 397 different geometric parameters during the 1<sup>st</sup> deformation stage; (a)  $\alpha$  and  $\theta_0$ ; (b) edge length

398  $a$ ,  $b$ ; (c) relative density  $\rho_r$ ; (d) height of rhombic honeycomb layer,  $h_2$ ;  $\alpha=45^\circ$ ,  $\theta_0=50^\circ$ ,  
399  $a=b=20\text{mm}$ ,  $h_2=15\text{mm}$ ,  $t=0.26\text{mm}$ ,  $x=2$ ,  $y=3$  unless otherwise noted;

400 The crushing responses of the 1<sup>st</sup> deformation stage of the proposed origami metamaterial with  
401 different geometric parameters are obtained from the analytical model of a unit column as  
402 shown in Figure 12. It should be noted that the unit column used for theoretical analysis is a  
403 unit cell of metamaterial rather than an actual column, as the metamaterial is very unlikely to  
404 be used in the form of a single column for energy absorption. Furthermore, friction contributed  
405 crushing force is not included as it is more prominent only for the structure with a large number  
406 of unit columns as shown previously in Figure 5 (c). Both face angle of the Miura-ori section  
407  $\alpha$  and the initial folding angle of  $\theta_0$  have significant influences on initial peak force, as shown  
408 in Figure 12 (a). With the increase in  $\alpha$  and decrease  $\theta_0$ , the initial peak force is reduced, where  
409 the crushing resistance at later stage is less affected. With the static angle  $\alpha$  closer to  $90^\circ$ , the  
410 Miura-ori section is closer to a square tube. With a larger initial folding angle  $\theta_0$ , the unit  
411 column becomes more slender and more difficult to bend along the horizontal creases. The  
412 crushable distance can be affected by these two angles as well. Furthermore, with the increase  
413 of edge length ratio of  $a/b$ , the initial peak force increases, while the crushable distance is solely  
414 dependent on  $b$ . Change of the layer height of rhombic honeycomb section,  $h_2$ , has minimal  
415 influence on the crushing response of the first stage of deformation, as the bending along  
416 vertical edges ( $h_2$ ) of rhombic honeycomb section is minimal during the first deformation stage.  
417 It should be noted that the relative density,  $\rho_r$ , and wall thickness,  $t$ , different from common  
418 cellular structures, are not directly related to the initial peak force. For instance, as shown in  
419 Figure 12 (c), the origami metamaterial with lower volumetric density or wall thickness could  
420 have a larger initial peak force. The crushing response for the proposed origami metamaterial  
421 is predominately dependent on the governing geometric parameters  $a$ ,  $b$ ,  $\alpha$ , and  $\theta_0$ . In general,  
422 the trend of the crushing resistance as well as the initial peak force is strongly depended on  
423  $\alpha$ , and  $\theta_0$ , and the mean crushing force is more affected by the side length  $a$ ,  $b$  and wall  
424 thickness  $t$ .

425 The ratio of initial peak to average crushing resistance is often used to measure the efficiency  
426 of an energy absorbing structure. This ratio is calculated from analytical model for a unit  
427 column with various geometric parameters as given in Figure 13. Similar to Figure 12, a smaller  
428 static angle  $\alpha$ , or a larger initial folding angle  $\theta_0$  leads to a smaller ratio of initial peak to  
429 average crushing resistance, indicating a more uniform crushing response and superior energy

430 absorbing performance. The value of both edge length,  $a$ ,  $b$  as well as the honeycomb layer  
 431 height,  $h_2$ , have less influence on the peak to average ratio. The structures with the peak to  
 432 average crushing force ratio less than 2 can be considered to be an ideal energy absorber, as  
 433 this ratio could reach 3 or 4 for some square columns [45, 53].



434

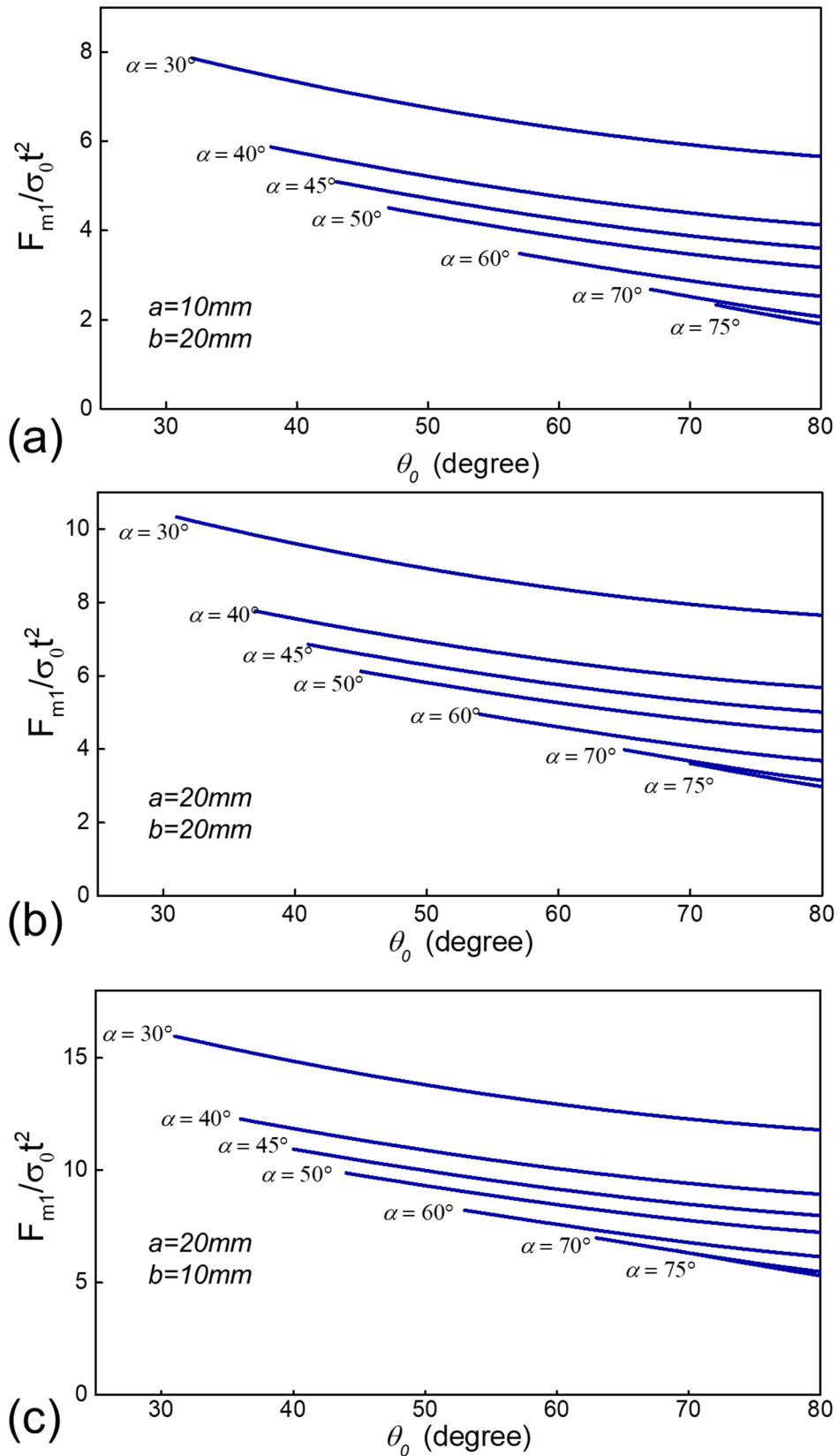
435 Figure 13. The ratio between initial peak ( $F_c$ ) and average crushing force ( $F_{m1}$ ) of a unit  
 436 column during the 1<sup>st</sup> stage of deformation with respect to  $\alpha$ , and  $\theta_0$  under different  
 437 configurations; (a)  $a=10$  mm,  $b=20$  mm,  $h_2=15$  mm; (b)  $a=20$  mm,  $b=20$  mm,  $h_2=15$  mm; (c)  
 438  $a=20$  mm,  $b=10$  mm,  $h_2=15$  mm; (d)  $a=20$  mm,  $b=20$  mm,  $h_2=20$  mm; (e)  $a=20$  mm,  $b=20$   
 439 mm,  $h_2=30$  mm;

## 440 5.2 Mean crushing force of the 1<sup>st</sup> and 2<sup>nd</sup> stages

441 The geometric parameters governing the mean crushing force of a unit column without the  
 442 friction contribution during the 1<sup>st</sup> deformation stage are shown in Figure 14. It should be noted  
 443 that only structures with uniform crushing response with peak to mean crushing force ratio less  
 444 than 2 are included in Figure 14. The static angle,  $\alpha$ , has the most significant influence on the  
 445 mean crushing force during the 1<sup>st</sup> stage of deformation. The mean crushing force can be three  
 446 times different with  $\alpha$  changed from 75° to 30°, while little changes in mean crushing force  
 447 can be observed with the change of  $\theta_0$ , for all three edge length ( $a, b$ ) configurations. A larger  
 448  $a/b$  may also lead to an increase in mean crushing force during the 1<sup>st</sup> stage of deformation.  
 449 However, a smaller edge length of  $b$  leads to a reduction in crushable distance from equation

450 (2), while the densification strain of the 1<sup>st</sup> stage of deformation,  $\varepsilon_{DI}$ , is determined only by  $a$ ,  
451  $t$ ,  $\alpha$ ,  $\theta_0$  according to equation (17) and (19). It should be noted that the connection bending  
452 resistance and the friction contributed crushing should be considered accordingly for the  
453 structures with multiple unit columns as expressed in equation (20) and (24)

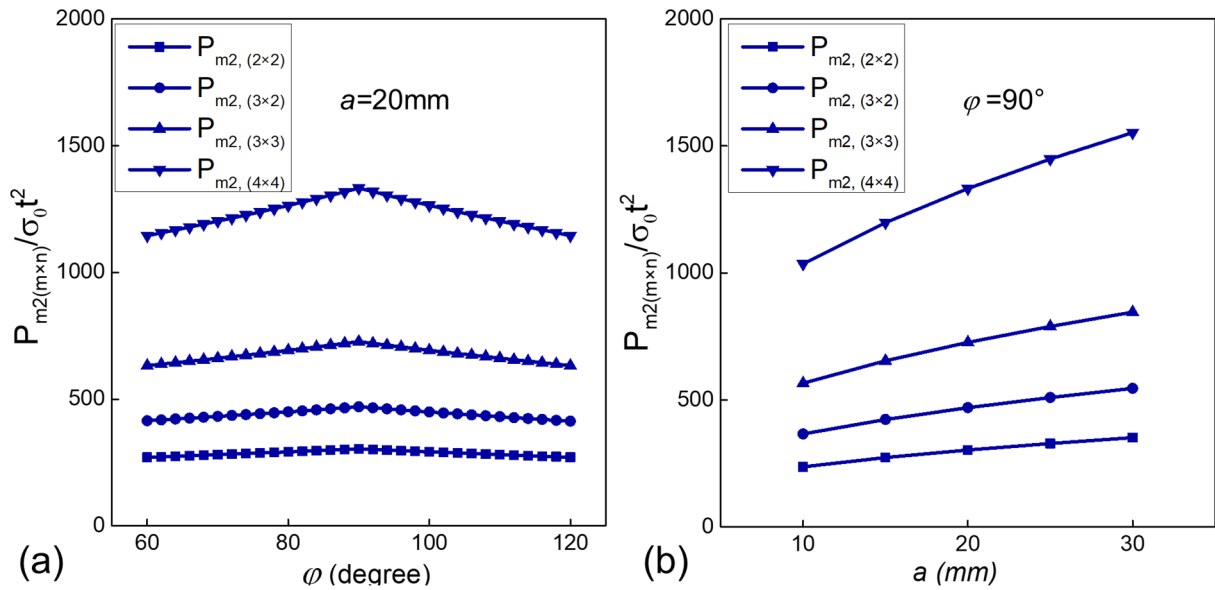




454

455 Figure 14. Mean crushing force of one unit column ( $F_{m1}$ ) during the 1<sup>st</sup> deformation stage  
 456 (without friction contribution) with respect to  $\alpha$ , and  $\theta_0$  under different edge lengths; (a)  
 457  $a=10\text{ mm}$ ,  $b=20\text{ mm}$ ; (b)  $a=20\text{ mm}$ ,  $b=20\text{ mm}$ ; (c)  $a=20\text{ mm}$ ,  $b=10\text{ mm}$ ;

458 Similar to honeycomb and multi-corner structures, the mean crushing force of the proposed  
 459 origami metamaterial during the 2<sup>nd</sup> stage of deformation is predominately dependent on the  
 460 cell size and wall thickness. As shown in Figure 15, with the increase of cell wall angle  $\varphi$  until  
 461 90°, the mean crushing force increases slightly, due to the symmetry of the dominating factor  
 462 of “X-shaped” intersection elements. For instance, an “X-shaped” intersection element with  
 463  $\varphi$  of 60° has the same mean crushing force as compared to that with  $\varphi$  of 120°, due to the  
 464 symmetry of the element. Furthermore, the increase of the unit columns number ( $m \times n$ ) within  
 465 a structure leads to a non-linear increase of the crushing force, due to the increase of the number  
 466 of “X-shaped” elements, especially when  $m, n$  is small. It should be noted that the mean  
 467 crushing force during the 2<sup>nd</sup> deformation stage is overestimated, as some localized buckling  
 468 deformation occurs at the initial state of the 2<sup>nd</sup> stage deformation on the cell wall due to the  
 469 off-set between rhombic layers. For the structures with  $b=L_f$ , the unit cells of rhombic  
 470 honeycomb layer stack directly on top of the other as the offset distance is equal to the length  
 471 of one unit cell at the beginning of the 2<sup>nd</sup> stage of deformation. The mean crushing force  
 472 predicted by the analytical model is likely to be more accurate for such structure, as each unit  
 473 cell on the rhombic honeycomb layer is supported by the lower layer.



474 (a) 475 Figure 15. Mean crushing force of structure with  $m \times n$  unit columns during the 2<sup>nd</sup> deformation  
 476 stage ( $P_{m2(m \times n)}$ ) with respect to different geometric parameters: (a)  $a=20$  mm; (b)  $\varphi=90^\circ$ ; Note:  
 477  $h_2=15\text{mm}$ ,  $t=0.26\text{mm}$

## 478 6 Conclusions

479 By combining Miura origami and rhombic honeycomb structure, a two-stage origami  
 480 metamaterial is proposed. The analytical model of the proposed structure has been developed

481 to predict the compressive properties for both deformation stages. The additional crushing force  
482 contributed by friction due to the equal and opposite Poisson's ratio in two in-plane directions  
483 during the 1<sup>st</sup> stage of deformation is included in the analytical model. Numerical simulation  
484 and quasi-static crushing tests of heat-treated specimens have been carried out. The analytical  
485 model is then verified with the numerical and quasi-static testing results. Good agreement of  
486 compressive properties is obtained between the analytical and numerical model, with slight  
487 discrepancies at the initial state of the 2<sup>nd</sup> deformation stage due to unit cell offset between  
488 rhombic honeycomb layers. The influence of geometric parameters on the compressive  
489 properties such as initial peak force, mean crushing force for both stages of deformation, has  
490 been investigated.

491 Different to common cellular structures, the relative density and wall thickness are not the sole  
492 governing parameters for the compressive properties of the structure. Other geometric  
493 parameters such as  $a$ ,  $b$ ,  $\alpha$ , and  $\theta_0$  affect the compressive response of the structure  
494 predominantly. The initial peak force of the 1<sup>st</sup> stage is mostly governed by the angles of  $\alpha$ , and  
495  $\theta_0$ , while the edge length,  $a$ ,  $b$  has more influence on the mean crushing force of the 1<sup>st</sup> stage.  
496 The additional crushing force attributed by friction during the 1<sup>st</sup> stage increases significantly  
497 with the increasing number of unit column and should not be neglected, due to the increases in  
498 swept area at the interfacial edges of the structure. For the 2<sup>nd</sup> stage of deformation, the  
499 compressive properties depend on the geometric parameters as well as the number of unit  
500 column within the structure. It is found that some of the undesired characteristics such as high  
501 initial peak crushing resistance can be mitigated by proper programming of the governing  
502 geometric parameters, without reducing the mean crushing resistance. A "graded" effect of  
503 crushing response with two uniform and programmable stages can be achieved without  
504 inducing initial peak force at either stage while keeping a uniform density throughout the  
505 structure.

## 506 **Acknowledgement**

507 The authors would like to thank Dr. Jianjun Zhang from Swinburne University for the  
508 discussion on the heat-treating of the pressed specimen. The financial support from the  
509 Australian Research Council via Discovery Early Career Researcher Award (DE160101116)  
510 is acknowledged.

511 **References**

- 512 [1] J.U. Surjadi, L. Gao, H. Du, X. Li, X. Xiong, N.X. Fang, Y. Lu, Mechanical Metamaterials  
513 and Their Engineering Applications, *Advanced Engineering Materials*, (2019).
- 514 [2] L.J. Gibson, M.F. Ashby, Cellular solids: structure and properties, Cambridge university  
515 press, 1999.
- 516 [3] A.G. Evans, J.W. Hutchinson, N.A. Fleck, M.F. Ashby, H.N.G. Wadley, The Topological  
517 Design of Multifunctional Cellular Metals, *Progress in Material Science*, 46 (2001) 309-327.
- 518 [4] V.S. Deshpande, M.F. Ashby, N.A. Fleck, Foam topology bending versus stretching  
519 dominated architectures, *Acta Materialia*, 49 (2001) 1035-1040.
- 520 [5] M.F. Ashby, A. Evans, N.A. Fleck, L.J. Gibson, J.W. Hutchinson, H.N.G. Wadley, Metal  
521 foams: a design guide, *Materials & Design*, 23 (2002) 119.
- 522 [6] T. Wierzbicki, Crushing analysis of metal honeycombs, *International Journal of Impact*  
523 *Engineering*, 1 (1983) 157-174.
- 524 [7] Z. Xue, J.W. Hutchinson, Crush dynamics of square honeycomb sandwich cores,  
525 *International Journal for Numerical Methods in Engineering*, 65 (2006) 2221-2245.
- 526 [8] H.N. Wadley, Multifunctional periodic cellular metals, *Philos Trans A Math Phys Eng Sci*,  
527 364 (2006) 31-68.
- 528 [9] J. Zhang, M. Ashby, The out-of-plane properties of honeycombs, *International Journal of*  
529 *Mechanical Sciences*, 34 (1992) 475-489.
- 530 [10] F. Côté, V.S. Deshpande, N.A. Fleck, A.G. Evans, The out-of-plane compressive behavior  
531 of metallic honeycombs, *Materials Science and Engineering: A*, 380 (2004) 272-280.
- 532 [11] R.M. Neville, F. Scarpa, A. Pirrera, Shape morphing Kirigami mechanical metamaterials,  
533 *Sci Rep*, 6 (2016) 31067.
- 534 [12] J. Zhang, G. Lu, D. Ruan, Z. Wang, Tensile behavior of an auxetic structure: Analytical  
535 modeling and finite element analysis, *International Journal of Mechanical Sciences*, 136 (2018)  
536 143-154.
- 537 [13] X. Ren, R. Das, P. Tran, T.D. Ngo, Y.M. Xie, Auxetic metamaterials and structures: A  
538 review, *Smart Materials and Structures*, (2018).
- 539 [14] H.Y. Jeong, S.C. An, I.C. Seo, E. Lee, S. Ha, N. Kim, Y.C. Jun, 3D printing of twisting  
540 and rotational bistable structures with tuning elements, *Sci Rep*, 9 (2019) 324.
- 541 [15] F. Pan, Y. Li, Z. Li, J. Yang, B. Liu, Y. Chen, 3D Pixel Mechanical Metamaterials, *Adv*  
542 *Mater*, 31 (2019) e1900548.
- 543 [16] H. Fang, K.W. Wang, S. Li, Asymmetric energy barrier and mechanical diode effect from  
544 folding multi-stable stacked-origami, *Extreme Mechanics Letters*, 17 (2017) 7-15.
- 545 [17] H. Fang, S.A. Chu, Y. Xia, K.W. Wang, Programmable Self-Locking Origami Mechanical  
546 Metamaterials, *Adv Mater*, 30 (2018) e1706311.
- 547 [18] J. Zhang, G. Lu, Dynamic tensile behaviour of re-entrant honeycombs, *International*  
548 *Journal of Impact Engineering*, 139 (2020).
- 549 [19] T. Fíla, P. Koudelka, P. Zlámal, J. Falta, M. Adorna, M. Neuhäuserová, J. Luksch, O.  
550 Jiroušek, Strain Dependency of Poisson's Ratio of SLS Printed Auxetic Lattices Subjected to

551 Quasi - Static and Dynamic Compressive Loading, *Advanced Engineering Materials*, (2019)  
552 1900204.

553 [20] N. Novak, L. Krstulović-Opara, Z. Ren, M. Vesenjak, Mechanical properties of hybrid  
554 metamaterial with auxetic chiral cellular structure and silicon filler, *Composite Structures*,  
555 (2019).

556 [21] X.-T. Wang, B. Wang, X.-W. Li, L. Ma, Mechanical properties of 3D re-entrant auxetic  
557 cellular structures, *International Journal of Mechanical Sciences*, 131-132 (2017) 396-407.

558 [22] A. Remennikov, D. Kalubadanage, T. Ngo, P. Mendis, G. Alici, A. Whittaker,  
559 Development and performance evaluation of large-scale auxetic protective systems for  
560 localised impulsive loads, *International Journal of Protective Structures*, (2019)  
561 204141961985808.

562 [23] G. Imbalzano, P. Tran, T.D. Ngo, P.V. Lee, Three-dimensional modelling of auxetic  
563 sandwich panels for localised impact resistance, *Journal of Sandwich Structures and Materials*,  
564 (2015).

565 [24] D. Xiao, X. Kang, Y. Li, W. Wu, J. Lu, G. Zhao, D. Fang, Insight into the negative  
566 Poisson's ratio effect of metallic auxetic reentrant honeycomb under dynamic compression,  
567 *Materials Science and Engineering: A*, 763 (2019) 138151.

568 [25] M. Schenk, S.D. Guest, Geometry of Miura-folded metamaterials, *Proc Natl Acad Sci U*  
569 *S A*, 110 (2013) 3276-3281.

570 [26] J. Zhang, D. Karagiozova, Z. You, Y. Chen, G. Lu, Quasi-static large deformation  
571 compressive behaviour of origami-based metamaterials, *International Journal of Mechanical*  
572 *Sciences*, 153 (2019) 194-207.

573 [27] J.M. Gattas, W. Wu, Z. You, Miura-Base Rigid Origami: Parameterizations of First-Level  
574 Derivative and Piecewise Geometries, *Journal of Mechanical Design*, 135 (2013) 111011-  
575 111011-111011.

576 [28] X. Zhou, H. Wang, Z. You, Mechanical properties of Miura-based folded cores under  
577 quasi-static loads, *Thin-Walled Structures*, 82 (2014) 296-310.

578 [29] X.M. Xiang, G. Lu, D. Ruan, Z. You, M. Zolghadr, Large deformation of an arc-Miura  
579 structure under quasi-static load, *Composite Structures*, 182 (2017) 209-222.

580 [30] Z. Li, W. Chen, H. Hao, Crushing behaviours of folded kirigami structure with square  
581 dome shape, *International Journal of Impact Engineering*, 115 (2018) 94-105.

582 [31] Z. Chen, T. Wu, G. Nian, Y. Shan, X. Liang, H. Jiang, S. Qu, Ron Resch Origami Pattern  
583 Inspired Energy Absorption Structures, *Journal of Applied Mechanics*, 86 (2018) 011005-  
584 011005-011007.

585 [32] G.V. Rodrigues, L.M. Fonseca, M.A. Savi, A. Paiva, Nonlinear dynamics of an adaptive  
586 origami-stent system, *International Journal of Mechanical Sciences*, 133 (2017) 303-318.

587 [33] D. Karagiozova, J. Zhang, G. Lu, Z. You, Dynamic in-plane compression of Miura-ori  
588 patterned metamaterials, *International Journal of Impact Engineering*, 129 (2019) 80-100.

589 [34] J. Ma, J. Song, Y. Chen, An origami-inspired structure with graded stiffness, *International*  
590 *Journal of Mechanical Sciences*, 136 (2018) 134-142.

591 [35] Z. Zhai, Y. Wang, H. Jiang, Origami-inspired, on-demand deployable and collapsible  
592 mechanical metamaterials with tunable stiffness, *Proceedings of the National Academy of*  
593 *Sciences*, 115 (2018) 2032-2037.

- 594 [36] E.T. Filipov, T. Tachi, G.H. Paulino, Origami tubes assembled into stiff, yet  
595 reconfigurable structures and metamaterials, *Proc Natl Acad Sci U S A*, 112 (2015) 12321-  
596 12326.
- 597 [37] L. de Waal, Z. You, Graded Origami Honeycomb Tube for Energy Absorption, in:  
598 International Design Engineering Technical Conferences and Computers and Information in  
599 Engineering Conference, American Society of Mechanical Engineers, 2019, pp.  
600 V05BT07A040.
- 601 [38] Y.L. He, P.W. Zhang, Z. You, Z.Q. Li, Z.H. Wang, X.F. Shu, Programming mechanical  
602 metamaterials using origami tessellations, *Composites Science and Technology*, 189 (2020).
- 603 [39] X. Zhang, H. Zhang, Theoretical and numerical investigation on the crush resistance of  
604 rhombic and kagome honeycombs, *Composite Structures*, 96 (2013) 143-152.
- 605 [40] N. Jones, *Structural impact*, Cambridge university press, 2011.
- 606 [41] W. Abramowicz, N. Jones, Dynamic axial crushing of square tubes, *International Journal*  
607 *of Impact Engineering*, 2 (1984) 179-208.
- 608 [42] W. Abramowicz, N. Jones, Dynamic progressive buckling of circular and square tubes,  
609 *International Journal of Impact Engineering*, 4 (1986) 243-270.
- 610 [43] W. Abramowicz, N. Jones, Transition from initial global bending to progressive buckling  
611 of tubes loaded statically and dynamically, *International Journal of Impact Engineering*, 19  
612 (1997) 415-437.
- 613 [44] W. Abramowicz, T. Wierzbicki, Axial Crushing of Multicorner Sheet Metal Columns,  
614 *Journal of Applied Mechanics*, 56 (1989) 113-120.
- 615 [45] W. Chen, T. Wierzbicki, Relative merits of single-cell, multi-cell and foam-filled thin-  
616 walled structures in energy absorption, *Thin-Walled Structures*, 39 (2001) 287-306.
- 617 [46] X. Zhang, G. Cheng, A comparative study of energy absorption characteristics of foam-  
618 filled and multi-cell square columns, *International Journal of Impact Engineering*, 34 (2007)  
619 1739-1752.
- 620 [47] A. Najafi, M. Rais-Rohani, Mechanics of axial plastic collapse in multi-cell, multi-corner  
621 crush tubes, *Thin-Walled Structures*, 49 (2011) 1-12.
- 622 [48] J.M. Gattas, Z. You, Quasi-static impact of indented foldcores, *International Journal of*  
623 *Impact Engineering*, 73 (2014) 15-29.
- 624 [49] R.K. Fathallah, J.M. Gattas, Z. You, Quasi-static crushing of eggbox, cube, and modified  
625 cube foldcore sandwich structures, *International Journal of Mechanical Sciences*, 101-102  
626 (2015) 421-428.
- 627 [50] G. Lu, T. Yu, *Energy Absorption of Structures and Materials*, Woodhead publishing  
628 limited, Cambridge England, 2003.
- 629 [51] F. Côté, V.S. Deshpande, N.A. Fleck, A.G. Evans, The compressive and shear responses  
630 of corrugated and diamond lattice materials, *International Journal of Solids and Structures*, 43  
631 (2006) 6220-6242.
- 632 [52] D. Ruan, G. Lu, F. Chen, E. Siores, Compressive behaviour of aluminium foams at low  
633 and medium strain rates, *Composite Structures*, 57 (2002) 331-336.
- 634 [53] G.M. Nagel, D.P. Thambiratnam, Computer simulation and energy absorption of tapered  
635 thin-walled rectangular tubes, *Thin-Walled Structures*, 43 (2005) 1225-1242.

